

B.E. PRODUCTION ENGINEERING SECOND YEAR FIRST SEMESTER  
SUPPLEMENTARY EXAM 2024

DEFORMATION OF SOLIDS

Time: 3 hours

Full Marks: 100

Answer any FIVE questions, taking any THREE  
from group-A and any TWO from group-B.

All parts of a question (a, b etc) should be answered at one place.  
All the dimensions in the figures are in centimeters unless indicated  
otherwise.

GROUP-A

- 1 (a) A rigid steel plate is supported by three concrete posts each having a  $10\text{cm} \times 10\text{cm}$  square cross-section as shown in Fig. 1(a). By accident, the middle post is  $0.05\text{cm}$  shorter than the other two before load  $P$  is applied. Find the safe value of load  $P$  if the working stress for the concrete in compression is  $200\text{kg/cm}^2$  and the modulus of elasticity  $E_c = 12 \times 10^4 \text{ kg/cm}^2$ .

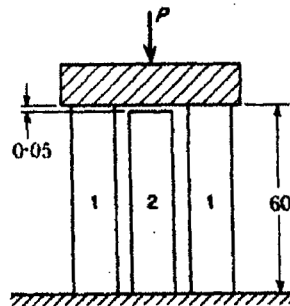


Fig.1 (a)

- (b) A steel tube 1 in. external diameter and  $3/4$  in. internal diameter encloses a copper rod  $5/8$  in. diameter to which it is rigidly joined at

each end. If, at a temperature of 60 °F there is no longitudinal stress, calculate the stress in the rod and tube when the temperature is raised to 400 °F. For steel  $E_s=30 \times 10^6$  lb/sq.in. and for copper  $E_c=14 \times 10^6$  lb/sq.in, coefficient of linear expansion for steel and copper is  $\alpha_s=6 \times 10^{-6}/^\circ\text{F}$  and  $\alpha_c=10 \times 10^{-6}/^\circ\text{F}$  respectively.

10+10

- 2(a) A solid truncated conical bar of circular cross-section tapers uniformly from a diameter  $d_1$  at its small end to  $d_2$  at the large end. The length of the bar is  $\ell$ . Determine the elongation due to an axial force  $P$  applied at each end.
- (b) For the simple structure shown in Fig. 2(b), member BC is a steel wire having diameter  $d = 3$  mm and member AB is a wood strut of 2.5cm-square cross-section. Calculate the horizontal and vertical components of the displacement of point B due to a vertical load  $P = 200$  kg acting as shown. For steel,  $E_s = 2 \times 10^6$  kg/cm<sup>2</sup> for wood,  $E_w = 10 \times 10^4$  kg/cm<sup>2</sup>.

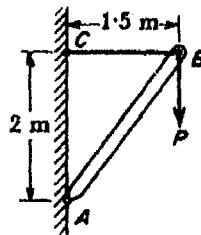


Fig.2 (b)

8+12

- 3(a) Consider a vertical bar of uniform cross-section with a flange at the lower end as shown in Fig. 3(a). Weight 'W' is released from a height 'h' and falls freely along the bar until it strikes the flange. Determine the maximum deflection ' $\delta$ ' of the bar and also calculate the maximum stress

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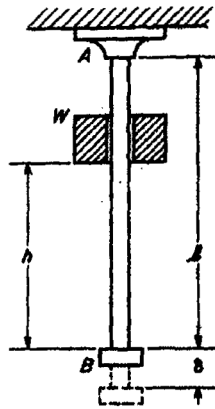


Fig. 3(a)

- (i) when 'h' is large compared to the elongation of the bar &  
(ii) when 'h' is zero.
- (b) A prismatic steel rod of length  $l$ , modulus of elasticity  $E$  and cross-sectional area  $A$  hangs vertically under its own weight. How much strain energy is stored in the bar if its weight per unit volume is  $\gamma$ ?

15+5

- 4(a) Show that bending stress can be expressed as

$$\sigma = My/I$$

Where each symbol holds its usual significance.

- (b) A shaft of diameter  $d$ , bent in the form of a semicircle AB of radius  $R$ , is built-in at A and loaded at B by a force  $P$  acting perpendicular to the plane of the ring as shown in Fig.4 (b). Thus any cross-section C of the ring is subjected to both bending and torsion. Assuming that  $d$  is small compared with  $R$  so that the theory of bending of straight bars may be used, find the value of  $\phi$  for which the principal stress  $\sigma$  will be a maximum.

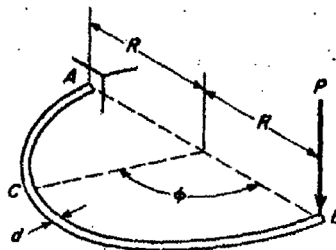


Fig.4 (b)

[ Turn over

10+10

- 5(a) Construct the complete shear force and bending moment diagram for the beam shown in Fig.5 (a). Also determine the maximum bending moment and the point of contra-flexure.

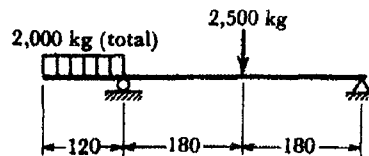


Fig.5 (a)

- (b) For a rectangular element shown in Fig. 5(b), the following numerical data are given  $\sigma_x = 100 \text{ kg/cm}^2$ ,  $\sigma_y = 75 \text{ kg/cm}^2$  and  $\tau_{xy} = 50 \text{ kg/cm}^2$ . Determine
- The values of  $\sigma_x$  and  $\tau$  on the plane whose normal is defined by  $\phi = 30^\circ$ .
  - The magnitude and direction of the principal stresses  $\sigma_1$  and  $\sigma_2$ .
  - The stress component and planes on which shear stress is the maximum.

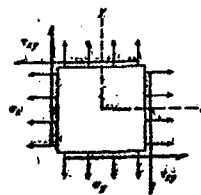


Fig.5 (b)

10+10

### GROUP-B

- 6(a) A hollow steel pipe is to be used as a standard to support a highway road sign as shown in Fig. 6(a). The maximum wind pressure on the face of the board is assumed to be  $250 \text{ kg/m}^2$ . The standard is unsupported laterally and its outside-to-inside diameter ratio is 1.12. The allowable working stress in shear is given as  $\tau_w = 600 \text{ kg/cm}^2$ . Calculate the required outside diameter  $d$  of the pipe.

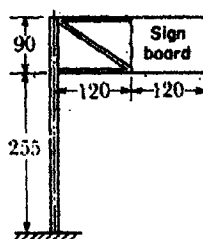


Fig. 6(a)

- (b) Show that the Euler's column formula for the case of a slender column built in at one end and free at the top is

$$P_{cr} = \frac{\pi^2 EI}{l^2}$$

Where the symbols have the usual meaning.

12+8

- 7(a) A torsion pendulum consists of a solid right circular disk suspended by a thin steel shaft of circular cross-section as shown in Fig. 7(a). The disk has weight  $W = 50$  kg; the shaft has length  $l = 50$  cm and diameter  $d = 3$  mm. For the shaft, allowable stresses in tension and shear, respectively, are  $\sigma_w = 1,120$  kg/cm<sup>2</sup> and  $\tau_w = 560$  kg/cm<sup>2</sup>. What is the maximum angle of twist  $\Phi$  that the shaft may have during torsional oscillations of the disk without exceeding either given working stress?

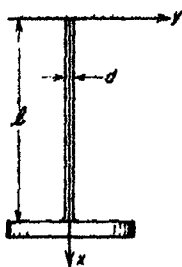


Fig. 7(a)

- (b) A truncated conical tank having the dimensions shown in Fig. 7(b) is filled with water ( $w = 1$  gm/cm<sup>3</sup>). Calculate the membrane stresses  $\sigma_1$  and  $\sigma_2$  for an element A of the wall situated as shown in the figure if  $t = 0.03$  cm.

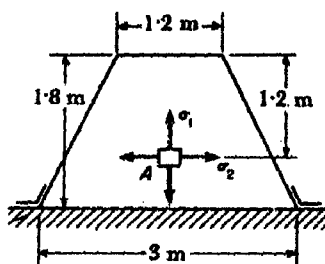


Fig. 7(b)

10+10

- 8(a) A simply supported beam is subjected to the loading as shown in Fig. 8(a). Determine the deflection at the point of application of the load

$P=10t$  and also find out the maximum deflection. The following numerical data are given:  $E = 2 \times 10^6 \text{ Kg/cm}^2$ ,  $I = 47 \times 10^3 \text{ cm}^4$ .

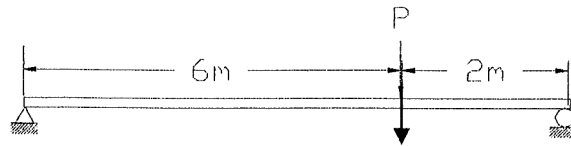


Fig. 8(a)

- (b) A solid steel shaft of diameter  $d = 6 \text{ mm}$  fits loosely inside a hollow steel shaft of inside diameter  $d = 6 \text{ mm}$  and outside diameter  $d_1 = 9 \text{ mm}$  as shown in Fig.8(b). A pin AA prevents relative rotation between the ends of the shafts at the left. Pinholes at the right are initially at right angles to each other as shown. The two shafts are now twisted in opposite directions until the pinholes at B line up and a pin BB is then inserted. How much strain energy will be locked in the system if  $l = 250 \text{ cm}$ ? Assume  $G = 84 \times 10^4 \text{ Kg/cm}^2$ .

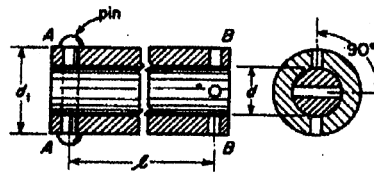


Fig. 8(b)