# Ref. No.: EX/PG/CNSE/T/112A/9/2017 EX/PG/ELE/T/112B/14/2017 MASTER OF ELECTRICAL ENGINEERING 1<sup>ST</sup> SEM. EXAMINATION-2017 MASTER OF CONTROL SYSTEM ENGINEERING 1<sup>ST</sup> SEM. EXAMINATION-2017 Subject: CONTROL SYSTEM ENGINEERING Time: Three Hours Full Marks: 100

Answer Any Five questions (5×20)

Question

Marks

4

No.

Q1 (a) Figure P-1(a) shows the block diagram of a d.c. motor control system with 2+4

$$G(s) = \frac{K(s+3)}{s(s+1)(s+2)}$$

The signal N(s) denotes the frictional torque at the motor shaft.



#### Figure P-1(a)

(i) Find the transfer function H(s) so that the output Y(s) is not affected by the disturbance torque N(s).

(ii) With H(s) as determined in (i), find the value of K so that the steady-state value of e(t) is 0.1 when the input is a unit-ramp function and N(s) = 0.

(b) Referring to the system shown in Figure P-1(b), determine the values of K and k such that the system has a damping ratio of 0.7 and an undamped natural frequency of 4 rad/sec.



3+7

4+2

10

(c) The block diagram of a control system is shown in Figure P-1(c) with

$$G(s) = \frac{100}{(1+0.1s)(1+0.5s)}$$

(i) Find the step-, ramp-, and parabolic-error constants.

(ii) Determine the minimum steady-state error that can be achieved with a unit-ramp input by varying the values of K and  $K_t$ .



### Figure P-1(c)

Q2 (a) Obtain the transfer function of the electrical network shown in Figure P-2(a).Determine whether this is an electrical lead network or lag network?



(b) In the system shown in Figure P-2(b), the numerical values of m, b, and k are given as m = 1 kg, b = 2 N-sec/m, and k = 100 N/m. The mass is displaced 0.05 m and released without initial velocity.

Figure P-2(a)

Find the frequency observed in the vibration. In addition, find the amplitude after four cycles.

The displacement x is measured from the equilibrium position.

Figure P-2(b)



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- (c) The unit impulse response of an LTI system is the unit step function u(t). 4 Find the response of the system to an excitation  $e^{-at}u(t)$ .
- Q3 (a) Consider the system shown in Figure P-3(a). Plot the root loci as the value of 10 k varies from 0 to  $\infty$ .



#### Figure P-3(a)

(b) Prove that a breakaway point  $\alpha$  satisfies the following:

$$\sum_{i=1}^{n} \frac{1}{\alpha + P_i} = \sum_{i=1}^{m} \frac{1}{\alpha + Z_i}$$

(c) Show that the root loci for a control system with

$$G(s) = \frac{K(s^2 + 6s + 10)}{s^2 + 2s + 10}, \ H(s) = 1$$

are arcs of the circle centered at the origin with radius equal to  $\sqrt{10}$ .

Q4 (a) The characteristic equation of a linear control system is given in the 8+4 following equation

$$s(s^{3} + 2s^{2} + s + 1) + K(s^{2} + s + 1) = 0$$

(i) Apply the Nyquist criterion to determine the values of K for system stability.

(ii) Verify the values of K by means of the Routh-Hurwitz criterion.

(b) The closed-loop frequency response of a second-order prototype system is shown in Figure P-4(b). Sketch the corresponding unit-step response of the system. Indicate the values of the maximum overshoot, peak time, and the steady-state error due to a unit-step input.



Q5 (a) Consider the unity-feedback control system whose open-loop transfer 6 function is

$$G(s) = \frac{\alpha s + 1}{s^2}$$

Determine the value of  $\alpha$  so that the phase margin is 45°.

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(b) Referring to the closed-loop system shown in Figure P-5(b), design a lead 14 compensator  $G_c(s)$  such that the phase margin is 45°, gain margin is not less than 8 dB, and the static velocity error constant  $K_v$  is 4 sec<sup>-1</sup>.



Figure P-5(b)

Q6 (a) Determine the values of K,  $T_1$ , and  $T_2$  of the system shown in Figure P-6(a) 14 so that the dominant closed-loop poles have the damping ratio of 0.5 and the undamped natural frequency of 3 rad/sec.



## Figure P-6(a)

(b) A control system with a PD controller is shown in Figure P-6(b). Find (analytically) the values of  $K_P$  and  $K_D$  so that the ramp-error constant  $K_v$  is 1000 and the damping ratio is 0.5.



Q7 (a) Find  $x_1(t)$  and  $x_2(t)$  of the system described by  $\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -3 & -2 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}$ 

where the initial conditions are  $x_1(0) = 1$  and  $x_2(0) = -1$ .

(b) Consider the system defined by

 $\dot{x} = Ax + Bu, y = Cx$ 

where

$$\boldsymbol{A} = \begin{bmatrix} 1 & 2 \\ -4 & -3 \end{bmatrix}, \quad \boldsymbol{B} = \begin{bmatrix} 1 \\ 2 \end{bmatrix}, \quad \boldsymbol{C} = \begin{bmatrix} 1 \\ 1 \end{bmatrix}$$

Transform the system equations into the controllable canonical form.

(c) Given the system in state equation form,  $\dot{x} = Ax + Bu$ , with

$$\boldsymbol{A} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & -3 & 0 \\ 0 & 0 & -2 \end{bmatrix}, \quad \boldsymbol{B} = \begin{bmatrix} 1 \\ 0 \\ 1 \end{bmatrix}$$

Can the system be stabilized by state feedback u = -Kx?

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Q8 (a) A controlled process is modeled by the following state equations

$$\frac{dx_1(t)}{dt} = x_1(t) - 2x_2(t), \qquad \frac{dx_2(t)}{dt} = 10x_1(t) - u(t)$$

The control u(t) is obtained from state feedback such that

$$u(t) = -k_1 x_1(t) - k_2 x_2(t)$$

Determine the region in the  $k_1$  -versus-  $k_2$  parameter plane in which the closed-loop system is asymptotically stable.

(b) Consider the system given by

$$\dot{x} = Ax + Bu$$

where

$$\boldsymbol{A} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -1 & -5 & -6 \end{bmatrix}, \quad \boldsymbol{B} = \begin{bmatrix} 0 \\ 1 \\ 1 \end{bmatrix}$$

By using the state-feedback control

$$u = -Kx$$

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it is desired to have the closed-loop poles at  $s = -2 \pm j4$  and s = -10. Determine the state-feedback gain matrix K. 14

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