

## Masters of Chemical Engineering Examination, 2017

(1<sup>st</sup> Semester)

## Advanced Heat Transfer

Time: Three Hours

Full Marks: 100

*Answer Question No. 1 and Any Two from the rest.*

[1] As a good model problem, we consider steady state heat transfer to fluid in steady flow through a tube. The fluid enters the tube at a temperature  $T_0$  and encounters a wall temperature at  $T_w$ , which can be larger or smaller than  $T_0$ . A simple version of this problem was first analyzed by Graetz (1883).

- Obtain the steady temperature distribution,  $T(r,z)$  in the fluid,
- Calculate the rate of heat transfer from the wall to the fluid.
- Sketch (qualitatively) the behavior of the Nusselt number as a function of dimensionless axial position.
- Explain the L ev eque approximation in connection to the Graetz problem.

The following assumptions may be adopted,

- Steady fully developed laminar flow; steady temperature field.
- Constant physical properties:  $\rho, \mu, k, C_p$ ; This assumption also implies incompressible Newtonian flow.
- Axisymmetric temperature field  $\Rightarrow \frac{\partial T}{\partial \phi}$ , where we are using the symbol  $\phi$  for the polar angle. Use the symbol  $\theta$  to represent dimensionless temperature.
- Negligible viscous dissipation.

[15+25+3+7=50]

[2] Starting with the following governing equation, develop a suitable model of heat transfer for laminar flow in a circular tube under hydro-dynamically and thermally developed conditions with Uniform Wall Temperature. At any point in the tube the boundary layer approximations may be applied. Here  $\alpha$  is the thermal diffusivity.

$$u \frac{\partial T}{\partial x} = \frac{\alpha}{r} \frac{\partial}{\partial r} \left( r \frac{\partial T}{\partial r} \right)$$

Hence show that, for a parabolic velocity profile,  $Nu_D = 3.656$

[25]

[3] Starting with the following governing equation, develop a suitable model of heat transfer for laminar flow in a circular tube under hydro-dynamically and thermally developed conditions with Uniform Wall Heat Flux. At any point in the tube the boundary layer approximations may be applied. Here  $\alpha$  is the thermal diffusivity.

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$$u \frac{\partial T}{\partial x} = \frac{\alpha}{r} \frac{\partial}{\partial r} \left( r \frac{\partial T}{\partial r} \right)$$

Hence show that, for a *parabolic* velocity profile, the fully developed temperature profile is:

$$T(r) = T_W + \frac{q_w r_0}{k} \left[ \left( \frac{r}{r_0} \right)^2 - \frac{1}{4} \left( \frac{r}{r_0} \right)^4 - \frac{7}{24} \right], k \text{ is the thermal conductivity.}$$

[25]

[4]

- a) Explain various regimes of Boiling with the help of a diagram. Explain the significance of *Peak Heat Flux* and *Leidenfrost point*.
- b) Develop an expression for the thickness of the boundary layer developed on a vertical flat plate during free convection heat transfer.

[7+18=25]

- [5] Consider film condensation on a vertical plate of length  $L$ . The plate temperature is maintained at  $T_w$ , and the vapour temperature at the edge of the film is the saturation temperature  $T_g$ . The film thickness is represented by  $\delta$ . It is assumed that the viscous shear of the vapour on the film is negligible. Obtain the following Nusselt number correlation.

$$Nu_x = \left[ \frac{\rho(\rho - \rho_v) g h_{fg} x^3}{4\mu_f k (T_g - T_w)} \right]^{1/4}$$

Where,  $h_{fg}$  is the latent heat of condensation and  $\mu_f$  is the viscosity of vapour respectively,  $k$  is the thermal conductivity of the material of the plate wall. Choose a coordinate system with positive direction of  $x$  being measured downward. Also find out a correlation for the average heat transfer coefficient.

[25]

## Continuity Equation

*Cylindrical*

$$\frac{\partial \rho}{\partial t} + \left\{ \frac{1}{r} \frac{\partial}{\partial r} (r \rho V_r) + \frac{1}{r} \frac{\partial}{\partial \theta} (\rho V_\theta) + \frac{\partial (\rho V_z)}{\partial z} \right\} = 0$$

## Navier Stokes Equations

*Cartesian*

$$x: \quad \rho \left\{ \frac{\partial V_x}{\partial t} + V_x \frac{\partial V_x}{\partial x} + V_y \frac{\partial V_x}{\partial y} + V_z \frac{\partial V_x}{\partial z} \right\} = \rho g_x - \frac{\partial p}{\partial x} + \mu \left\{ \frac{\partial^2 V_x}{\partial x^2} + \frac{\partial^2 V_x}{\partial y^2} + \frac{\partial^2 V_x}{\partial z^2} \right\}$$

$$y: \quad \rho \left\{ \frac{\partial V_y}{\partial t} + V_x \frac{\partial V_y}{\partial x} + V_y \frac{\partial V_y}{\partial y} + V_z \frac{\partial V_y}{\partial z} \right\} = \rho g_y - \frac{\partial p}{\partial y} + \mu \left\{ \frac{\partial^2 V_y}{\partial x^2} + \frac{\partial^2 V_y}{\partial y^2} + \frac{\partial^2 V_y}{\partial z^2} \right\}$$

$$z: \quad \rho \left\{ \frac{\partial V_z}{\partial t} + V_x \frac{\partial V_z}{\partial x} + V_y \frac{\partial V_z}{\partial y} + V_z \frac{\partial V_z}{\partial z} \right\} = \rho g_z - \frac{\partial p}{\partial z} + \mu \left\{ \frac{\partial^2 V_z}{\partial x^2} + \frac{\partial^2 V_z}{\partial y^2} + \frac{\partial^2 V_z}{\partial z^2} \right\}$$

*Cylindrical*

*r:*

$$\begin{aligned} \rho \left\{ \frac{\partial V_r}{\partial t} + V_r \frac{\partial V_r}{\partial r} + \frac{V_\theta}{r} \frac{\partial V_r}{\partial \theta} - \frac{V_\theta^2}{r} + V_z \frac{\partial V_r}{\partial z} \right\} \\ = \rho g_r - \frac{\partial p}{\partial r} + \mu \left\{ \frac{\partial}{\partial r} \left( \frac{1}{r} \frac{\partial}{\partial r} (r V_r) \right) + \frac{1}{r^2} \frac{\partial^2 V_r}{\partial \theta^2} - \frac{2}{r^2} \frac{\partial V_\theta}{\partial \theta} + \frac{\partial^2 V_r}{\partial z^2} \right\} \end{aligned}$$

*\theta:*

$$\begin{aligned} \rho \left\{ \frac{\partial V_\theta}{\partial t} + V_r \frac{\partial V_\theta}{\partial r} + \frac{V_\theta}{r} \frac{\partial V_\theta}{\partial \theta} + \frac{V_r V_\theta}{r} + V_z \frac{\partial V_\theta}{\partial z} \right\} \\ = \rho g_\theta - \frac{1}{r} \frac{\partial p}{\partial \theta} + \mu \left\{ \frac{\partial}{\partial r} \left( \frac{1}{r} \frac{\partial}{\partial r} (r V_\theta) \right) + \frac{1}{r^2} \frac{\partial^2 V_\theta}{\partial \theta^2} + \frac{2}{r^2} \frac{\partial V_r}{\partial \theta} + \frac{\partial^2 V_\theta}{\partial z^2} \right\} \end{aligned}$$

*z:*

$$\rho \left\{ \frac{\partial V_z}{\partial t} + V_r \frac{\partial V_z}{\partial r} + \frac{V_\theta}{r} \frac{\partial V_z}{\partial \theta} + V_z \frac{\partial V_z}{\partial z} \right\} = \rho g_z - \frac{\partial p}{\partial z} + \mu \left\{ \frac{1}{r} \frac{\partial}{\partial r} \left( r \frac{\partial V_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 V_z}{\partial \theta^2} + \frac{\partial^2 V_z}{\partial z^2} \right\}$$

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