B.E. CIVIL ENGINEERING SECOND YEAR SECOND SEMESTER (Old) - 2017

Subject: THERMODYNAMICS & HEAT POWER

Time: Three hours Full Marks: 100

Answer any <u>five</u> questions
Use of thermodynamic tables and charts is permitted
Assume any suitable data if necessary
All parts of the same question must be answered together
Symbols carry usual meaning

- a) State and explain the zeroth law of thermodynamics in brief.
 b) State the similarities and differences between heat and work.
- c) A closed system of a gas of 5kg mass undergoes expansion process (pv^{1.3}=constant) from 1Mpa and 0.5m³ to 0.5Mpa. The specific internal energy of the gas is given by, u=1.8pv + 85 kJ/kg, where p in kPa, v in m³/kg. Determine heat and work interactions and change in internal energy.
- 2. a) A gas undergoes a thermodynamic cycle consisting of the following processes: Process 1-2: constant pressure, $p_1=1.4$ bar, $V_1=0.028m^3$, $W_{1-2}=10.5kJ$.
- Process 2-3: compression with pv=constant, $U_3=U_2$.
- Process 3-1: constant volume, U_1 - U_3 = -26.4kJ.

There are no changes in kinetic and potential energies. a) sketch the cycle on the p-v diagram. b) calculate the net work of the cycle in kJ. c) calculate the heat transfer for process 1-2 in kJ.

- b) A compressor receives air at 100kPa, 25^{0}C and discharges that at 185^{0}C , 400kPa. The velocity of air at the inlet and exit are 150m/s and 50m/s respectively. The power input to the compressor is 3000kW. Neglecting heat transfer from the compressor and change in potential energy, find out the mass flow rate through the compressor. Assume $c_{p}=1.005\text{kJ/kgK}$.
- 3. a) Draw the p-v-T surface for pure water. Show on the surface an isobaric process line. Show the critical point and the triple line on this surface.

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- b) Define and explain dryness fraction of steam.
- c) A vessel having a volume of 0.05m^3 contains a mixture of saturated steam and water at a temperature of 300°C . The mass of liquid water is 10kg. Determine the pressure, mass, enthalpy, and entropy of total contents present in the vessel.

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- 4. a) Show that no heat engine can have higher efficiency than a reversible heat engine operating between two given thermal reservoirs.
 - b) A reversible heat engine operates between three heat reservoirs A, B and C. The heat received by the engine from each of the reservoirs A and B is same and at temperatures TA and TB respectively; and the engine rejects heat to the reservoir C at temperature T_C. If the engine efficiency is K times the efficiency of a reversible heat engine operating between two reservoirs A and C only, show that:

$$\frac{T_A}{T_B} = 2(1 - K)\frac{T_A}{T_C} + (2K - 1)$$

- 10 5. a) Show that entropy is a property of a system.
- b) Deduce the expression for change of entropy of an ideal gas during a reversible 05 process in terms of initial and final pressure and volume.
- c) Show the expression of heat transfer for a reversible polytropic process and hence find 05 the expression of polytropic specific heat.
- 6) a) Derive the expression of the air standard efficiency of an Otto cycle as a function of compression ratio.
- b) An air standard diesel cycle has a compression ratio of 18 and the heat transferred to the working fluid per cycle is 1800kJ/kg. At the beginning of the compression process the pressure is 0.1 MPa and the temperature is 15°C. Determine: i) the pressure and temperature at each cardinal point in the cycle. ii) thermal efficiency and iii) the mean effective pressure.
- 05x4=207. Write short notes on the following:
 - a) Reversible process b) Assumptions of air standard cycles c) Carnot cycle d) throttling process