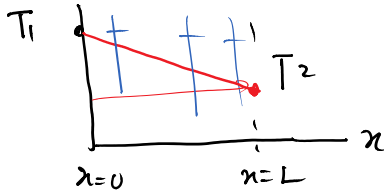
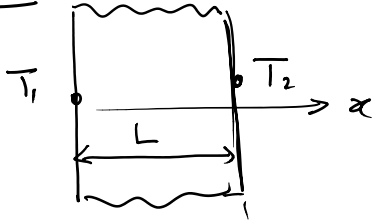


1-D Steady State Heat Conduction (k=const)

$$\frac{\partial}{\partial t}(\rho c T) = \frac{1}{r^n} \frac{\partial}{\partial r} \left(r^n \frac{\partial T}{\partial r} \right) + \dot{q}'''$$

1-D Cartesian System with NO heat Generation w/

BC1



$$0 = \frac{d}{dx} \left(\frac{dT}{dx} \right) + 0$$

$$\frac{d^2 T}{dx^2} = 0$$

$$\Rightarrow T = C_1 x + C_2 \quad \text{--- (1)}$$

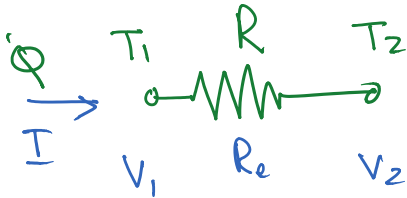
BC1 \Rightarrow

$$\left. \begin{aligned} x=0 &\Rightarrow T=T_1 \\ x=L &\Rightarrow T=T_2 \end{aligned} \right\} \begin{aligned} C_2 &= T_1 \\ C_1 &= \frac{T_2 - T_1}{L} \end{aligned}$$

$$T(x) = \frac{(T_2 - T_1)x}{L} + T_1$$

$$T(x) = T_1 - \frac{T_1 - T_2}{L} x \quad \text{--- (2)}$$

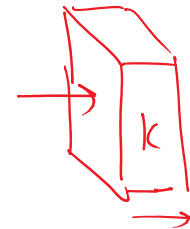
$$\dot{Q} = -kA \frac{dT}{dx} \Big|_x = -kA C_1 = kA \frac{T_1 - T_2}{L}$$



$$\dot{Q} = \frac{T_1 - T_2}{\left(\frac{L}{kA} \right)} \quad \text{thermal resistance}$$

$$\rightarrow I = \frac{V_1 - V_2}{R_{\parallel}} \quad \text{OHM'S LAW}$$

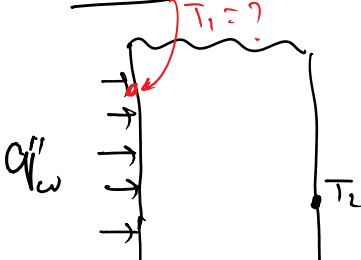
Thermal resistance $R = \frac{L}{kA}$



Thermal Resistance analog is

- valid ONLY \rightarrow
- ① 1-D
 - ② steady
 - ③ No heat generation ($\dot{q}'''=0$)
 - ④ $k = \text{const.}$

BC2

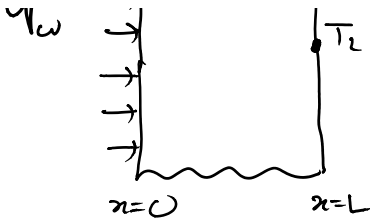


at $x=0$, $-k \frac{dT}{dx} = q_w''$

$x=L$ $T = T_2$

$$T = C_1 x + C_2$$

$$-k \frac{dT}{dx} = -k C_1 = q_w'' \Rightarrow C_1 = -\frac{q_w''}{k}$$



$$T = C_1 x + C_2$$

$$-\frac{k \frac{dT}{dx}}{dx} = -k C_1 = q''_w \Rightarrow C_1 = -\frac{q''_w}{k}$$

$$T_2 = C_1 L + C_2 \Rightarrow C_2 = T_2 + \frac{q''_w}{k} L$$

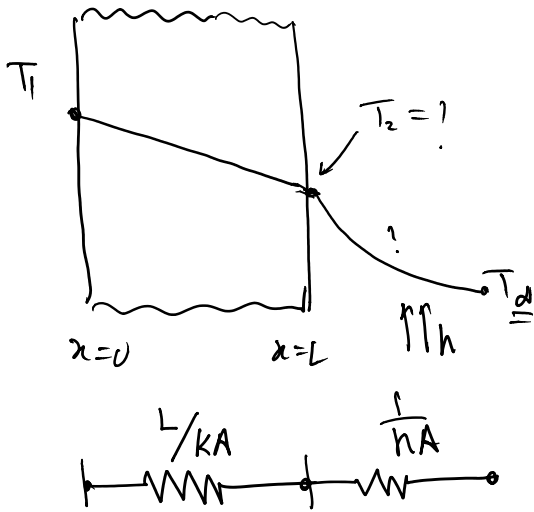
$$T(x) = T_2 + \frac{q''_w}{k} (L-x)$$

at $x=0$, $T_1 = T_2 + \frac{q''_w L}{k}$

$$A k \frac{T_1 - T_2}{L} = q''_w A = \dot{Q}$$

$$\dot{Q} = \frac{T_1 - T_2}{L/kA} = \frac{\Delta T}{R}$$

BC 3

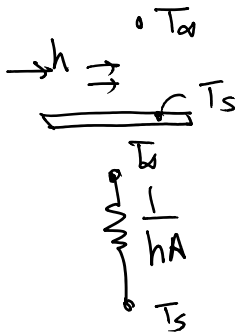


$$T(x) = T_1 - \frac{hx}{k+hL} (T_1 - T_\infty)$$

$$T_2 = \frac{kT_1 + hL T_\infty}{k+hL}$$

$$\dot{Q} = q''_w A = \frac{T_1 - T_\infty}{\frac{1}{hA} + \frac{L}{kA}}$$

Convection resistance
Conduction resistance



$$q''_w = h (T_s - T_\infty) \quad \text{W/m}^2$$

$$\dot{Q} = hA (T_s - T_\infty) \quad \text{W}$$

$$= \frac{T_s - T_\infty}{\frac{1}{hA}}$$

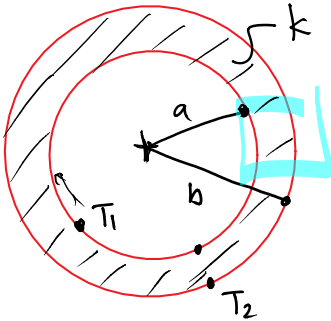
$$R_{conv} = \frac{1}{hA}$$

① we can express the conduction HT. in terms of a resistance network if it is 1-D, steady, heat transfer with $k = \text{const}$ & $q''_w = 0$

$$R_{cond} = \frac{L}{kA} \quad \left[\frac{k}{W} \right] \quad \frac{\frac{m}{m^2} K}{\frac{W}{m^2}}$$

$$\left\{ \begin{array}{l} R_{\text{cond}} = \frac{L}{kA} \quad \left[\frac{\text{m}^2}{\text{W}} \right] \\ R_{\text{conv}} = \frac{1}{hA} \quad \left[\frac{\text{K}}{\text{W}} \right] \end{array} \right. \quad \begin{array}{l} \text{W m}^2 \\ \frac{\text{m}^2 \text{K}}{\text{W m}^2} \end{array}$$

steady
1-D, Heat Transfer across a hollow cylinder with $k = \text{const}$, $\dot{q}''' = 0$



$$\text{GIDE} \quad \frac{1}{r} \frac{d}{dr} \left(r \frac{dT}{dr} \right) = 0 \quad \text{for } [a \leq r \leq b]$$

or $\frac{d}{dr} \left(r \frac{dT}{dr} \right) = 0$

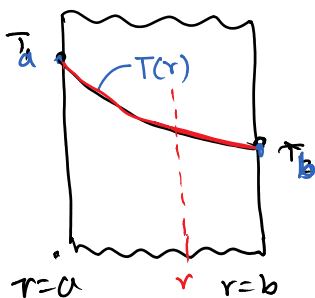
Integrate once w.r.t. r

$$r \frac{dT}{dr} = C_1 \quad \text{--- (2)}$$

Cannot impose the B.C. at $r=0$

Integrating once more

$$T = C_1 \ln r + C_2 \quad \text{--- (3)}$$



BC at $r=a$, $T = T_a$
at $r=b$, $T = T_b$

Apply this in (3) \Rightarrow

$$\left. \begin{array}{l} T_a = C_1 \ln a + C_2 \\ T_b = C_1 \ln b + C_2 \end{array} \right\} \quad \text{--- (4)}$$

From (4)

$$C_1 = \frac{T_a - T_b}{\ln a - \ln b} = \frac{T_b - T_a}{\ln b/a}$$

$$C_2 = \left[T_a - \frac{T_b - T_a}{\ln b/a} \ln a \right]$$

Using C_1 & C_2 in eqn (3)

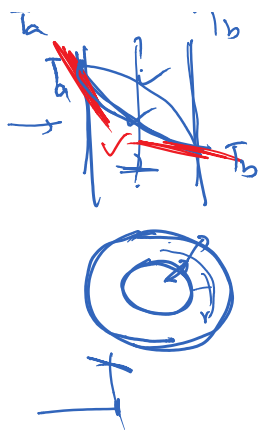
$$T(r) = C_1 \ln r + C_2$$

$$= \frac{T_b - T_a}{\ln b/a} \cdot \ln r + T_a - \frac{T_b - T_a}{\ln b/a} \ln a$$

$$T(r) - T_a = \frac{T_b - T_a}{\ln b/a} \ln r/a$$

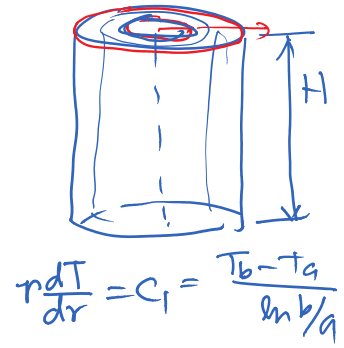
$$\text{or } \boxed{\frac{T(r) - T_a}{T_b - T_a} = \frac{\ln r/a}{\ln b/a}} \quad \text{--- (5)}$$





$$T_b - T_a \quad \ln b/a$$

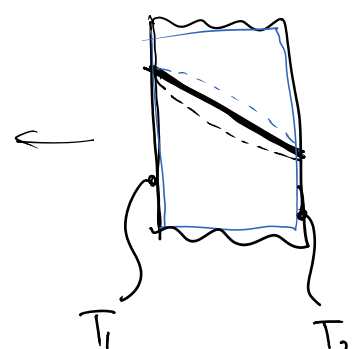
$$\begin{aligned} \dot{Q} &= A \times q'' = 2\pi r H \times \left(-k \frac{dT}{dr} \right)_r \\ &= -2\pi H k \left(r \frac{dT}{dr} \right)_r \\ &= 2\pi H k \frac{T_b - T_a}{\ln b/a} \end{aligned}$$



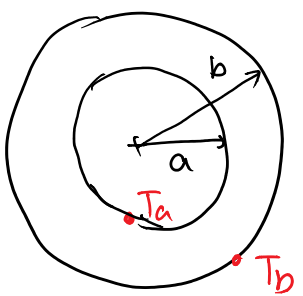
$$\dot{Q} = \frac{T_a - T_b}{\left(\frac{\ln b/a}{2\pi k H} \right)} = \frac{T_a - T_b}{R}$$

Thermal resistance for a cylindrical shell

$$R = \frac{\ln b/a}{2\pi k H} \quad \left[\frac{mK}{Wm} \right]$$



1-D, steady heat conduction for a spherical shell without q'' & with k=const



GDE $\frac{1}{r^2} \frac{d}{dr} \left(r^2 \frac{dT}{dr} \right) = 0$
 $\Rightarrow \frac{d}{dr} \left(r^2 \frac{dT}{dr} \right) = 0$
 $\Rightarrow r^2 \frac{dT}{dr} = C_1$

$$\Rightarrow T(r) = -\frac{C_1}{r} + C_2 \quad \text{--- (1)}$$

B.C.

$$\begin{aligned} T(a) &= T_a \\ T(b) &= T_b \end{aligned}$$

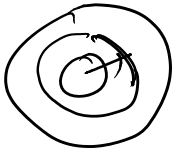
$$\left. \begin{aligned} T_a &= -\frac{C_1}{a} + C_2 \\ T_b &= -\frac{C_1}{b} + C_2 \end{aligned} \right\} \text{(2)}$$

$$\left. \begin{aligned} C_1 &= \frac{ab}{b-a} (T_a - T_b) \\ C_2 &= \frac{bT_b - aT_a}{b-a} \end{aligned} \right\} \text{(3)}$$

~~I was missing this -ve sign~~

→ $C_2 = \frac{b \ln a - a \ln b}{b-a}$

$$T(r) = \frac{a}{r} \left(\frac{b-r}{b-a} \right) \cdot T_a + \frac{b}{r} \cdot \left(\frac{r-a}{a-b} \right) T_b \quad (4)$$



$$\dot{Q} = -kA \times \frac{dT}{dr} \Big|_r = -k \cdot 4\pi r^2 \frac{dT}{dr} \Big|_r$$

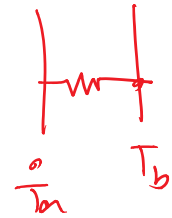
$$\dot{Q} = -k \times 4\pi C_1$$

$$= +4\pi k \frac{(T_a - T_b)}{\left(\frac{b-a}{ab}\right)} = \frac{T_a - T_b}{\frac{1}{4\pi k} \left[\frac{1}{a} - \frac{1}{b} \right]}$$

$$\dot{Q} = \frac{T_a - T_b}{R}$$

$$R = \frac{1}{4\pi k} \left[\frac{1}{a} - \frac{1}{b} \right]$$

$$\left[\frac{m^2 K}{W m} \right]$$



Thermal resistances for

- ① Plane wall of thickness L

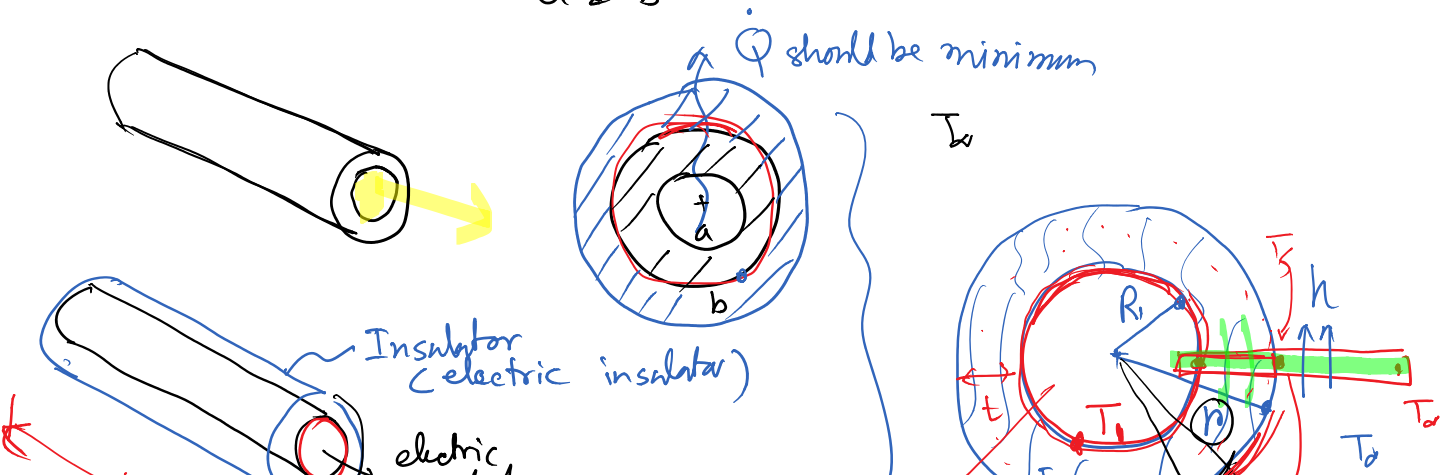
$$R_{\text{plane}} = \frac{L}{kA}$$

- ② Cylindrical shell of height H and inner & outer radii of a & b

$$R_{\text{cyl.}} = \frac{\ln b/a}{2\pi k H}$$

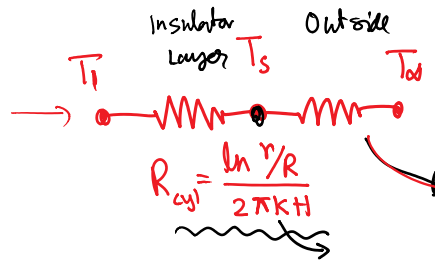
- ③ Spherical shell of inner & outer radii a & b

$$R_{\text{sph}} = \frac{1}{4\pi k} \left(\frac{1}{a} - \frac{1}{b} \right)$$





electric conduction

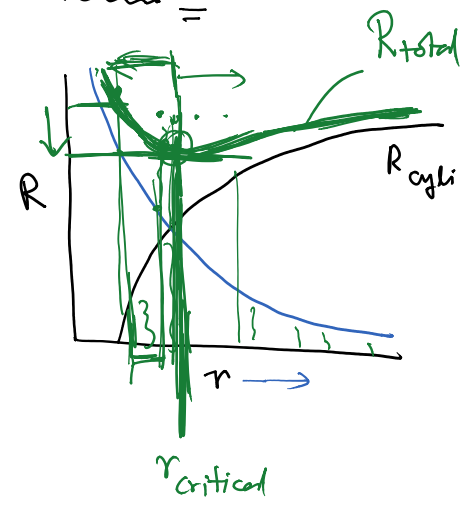


$$R_{cyl} = \frac{\ln r/R}{2\pi kH}$$

$$R_{conv} = \frac{1}{hA} = \frac{1}{2\pi r H h}$$

$$R_{tot} = R_{cyl} + R_{conv}$$

$$R_{tot} = \frac{\ln r/R}{2\pi kH} + \frac{1}{2\pi H h r}$$



$$\frac{dR_{tot}}{dr} = \frac{1}{r} \frac{1}{2\pi kH} - \frac{1}{2\pi H h r^2}$$

$$\frac{d^2R_{tot}}{dr^2} = -\frac{1}{r^2} \frac{1}{2\pi kH} + \frac{1}{\pi H h r^3}$$

For extreme value of R_{tot}

$$\frac{dR_{tot}}{dr} = 0 \Rightarrow \frac{1}{r} \frac{1}{2\pi kH} = \frac{1}{2\pi H h} \frac{1}{r^2}$$

$$\Rightarrow \boxed{r_{crit} = \frac{k}{h}}$$

$$\left[\frac{m}{mk} \cdot \frac{m^2}{h} \right]$$

put $r = k/h$

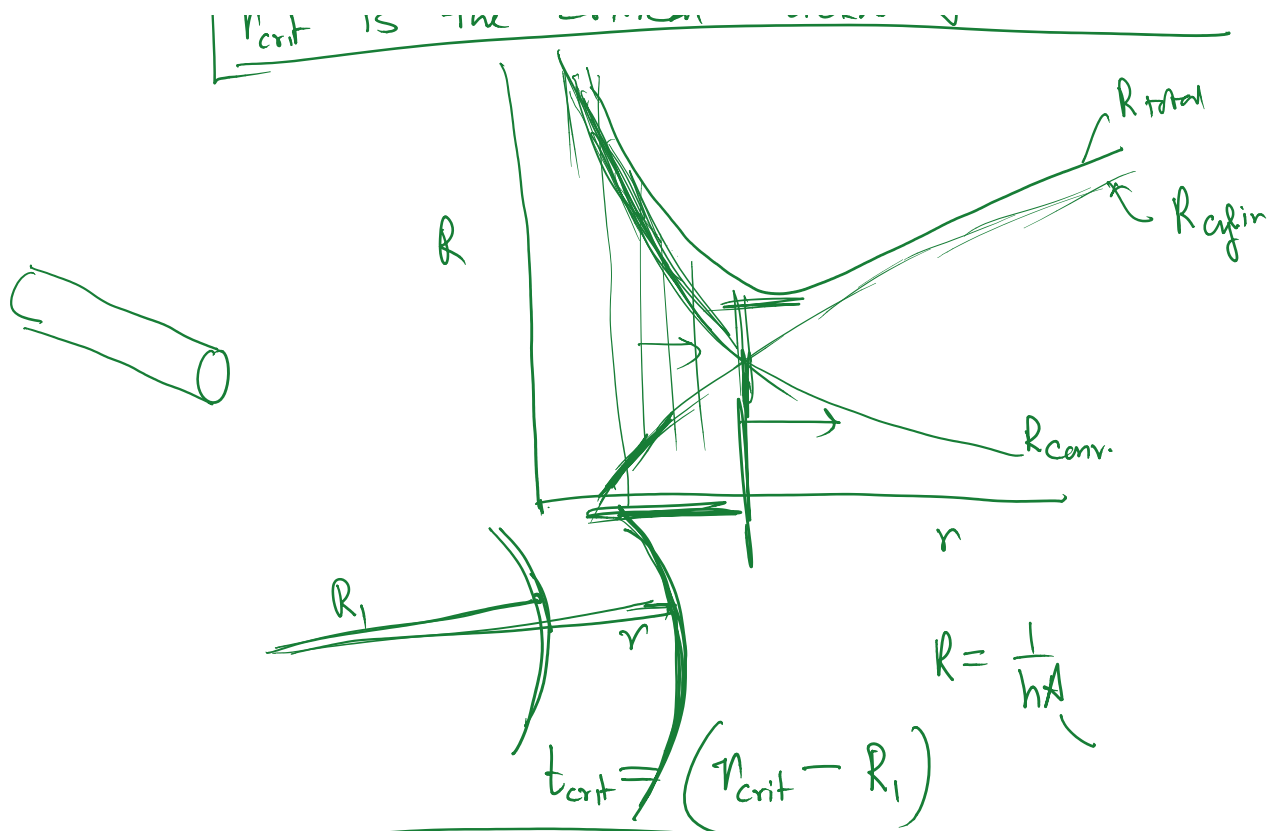
$$\frac{d^2R_{tot}}{dr^2} = -\frac{h^2}{k^2} \times \frac{1}{2\pi kH} + \frac{1}{\pi H h} \frac{h^3}{k^3}$$

$$= + \frac{1}{2\pi k^3 H} \quad (+ve)$$

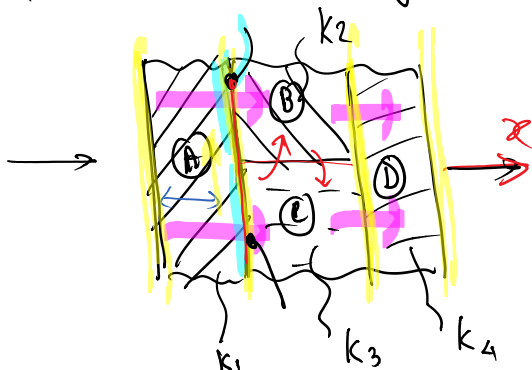
R_{tot} is minimum when

$$\boxed{r = \frac{k}{h}}$$

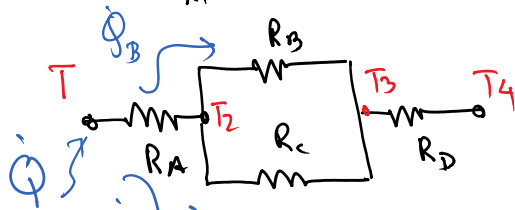
r_{crit} is the critical thickness of insulation



Thermal resistance by a composite wall



- ① Heat Transfer is 1D, steady
- ② No heat generation
- ③ Materials have different k but they are homogeneous
- ④ all surfaces \perp to the heat transfer are isothermal



$L_c = L_B$

$R_A = \frac{L_A}{k_A A_A}$

$R_B = \frac{L_B}{k_B A_B}$

$R_D = \frac{L_D}{k_D A_D}$

$R_C = \frac{L_C}{k_C A_C}$

$\dot{Q}_B = \frac{T_3 - T_2}{R_B}$

$(\dot{Q}_B + \dot{Q}_C) = \dot{Q}_A = \dot{Q}_D$

$$\dot{Q}_c = \frac{T_3 - T_2}{R_c}$$