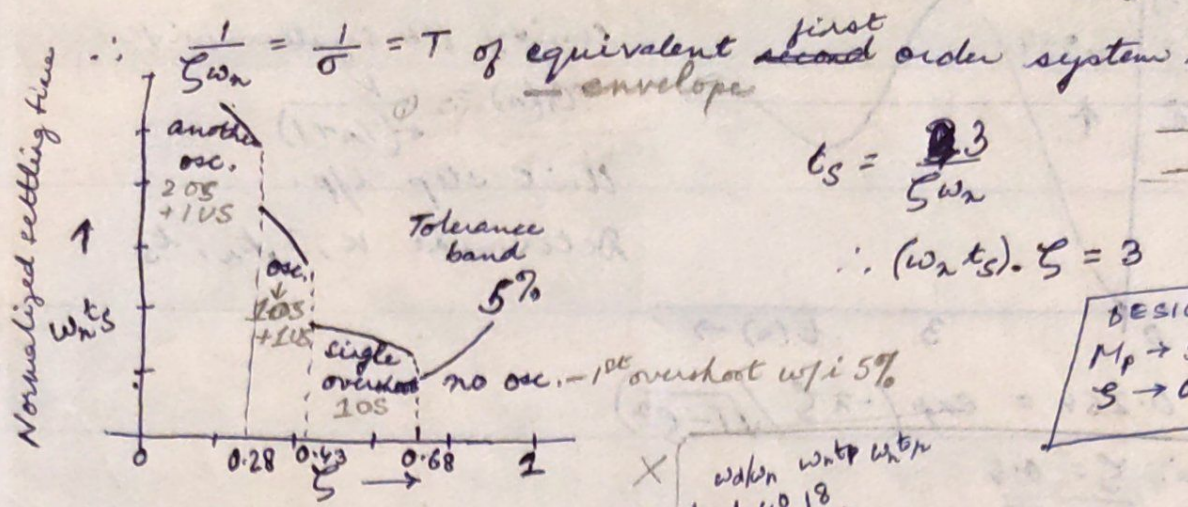


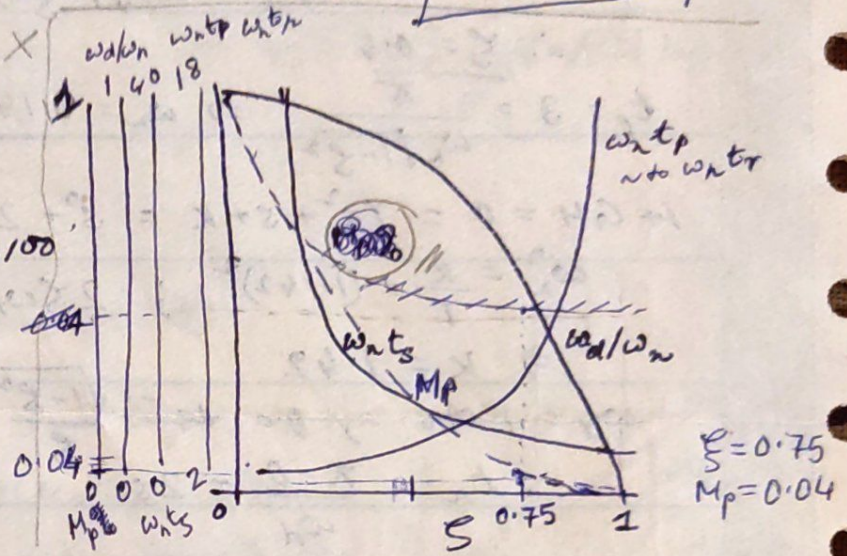
fig with 20% major  
 $\frac{p}{100} = e^{-\zeta \omega_n t_s}$   
 $I$



DESIGN  
 $M_p \rightarrow 5-25\%$   
 $\zeta \rightarrow 0.4-0.7$

$\zeta = 1/\sqrt{2} = 0.707$   
 $M_p = 4.3\%$   
 $\zeta = 0.68$   
 $M_p = 5.4\%$   
 $\zeta = 0.7$   
 $M_p = 4.6\%$   
 $\zeta = 0.9$   
 $M_p = 0.15\%$

$\therefore \omega_n t_p = \frac{\pi}{\sqrt{1-\zeta^2}}$   
 $M_p(\%) = e^{(-\pi \zeta / \sqrt{1-\zeta^2})} \times 100$   
 $\omega_n t_s = \frac{3}{\zeta}$   
 $\frac{\omega_d}{\omega_n} = \sqrt{1-\zeta^2}$   
 $\Rightarrow \left(\frac{\ln M_p}{\pi}\right)^2 (1-\zeta^2) = \zeta^2$



Usually  $M_p$  &  $t_s$  specified  $\rightarrow$  significance in relv. to freq. responses.

Note: (i)  $t_d = 0.5 t_{p1} = 0.5 t_r$  ( $t_r$  being for 0-100%)

(ii)  $t_r = \frac{\pi - \theta}{\omega_d}$       $\theta = \cos^{-1} \zeta \Rightarrow \omega_n t_r = \frac{\pi - \theta}{\sqrt{1-\zeta^2}}$

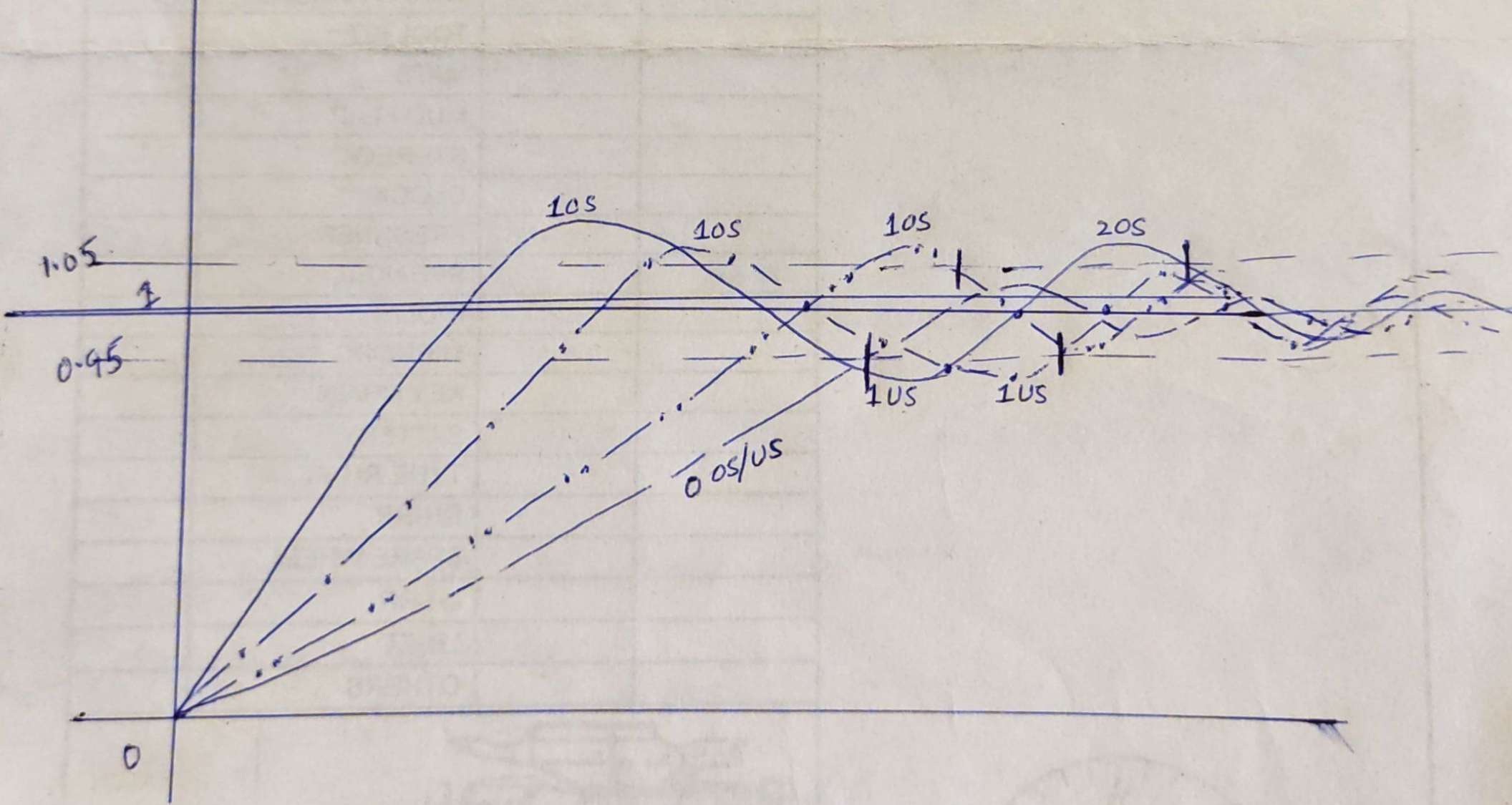
$\therefore \zeta \uparrow \Rightarrow t_d, t_r \uparrow$ ;  $t_s$  is large for small & large  $\zeta$  (beyond 0.707)  
 $M_p \downarrow$  ( $0 < \zeta < 1$ )

$\therefore$  optimum  $\zeta = 0.707$ :  $M_p = 4.3\%$  while  $t_d, t_r, t_s$  reasonable.

$\zeta$  fixed,  $\omega_n \uparrow$ ,  $M_p = c$ ,  $t_d, t_r, t_s, t_p \downarrow$

$\therefore$  desire high  $\omega_n$ .

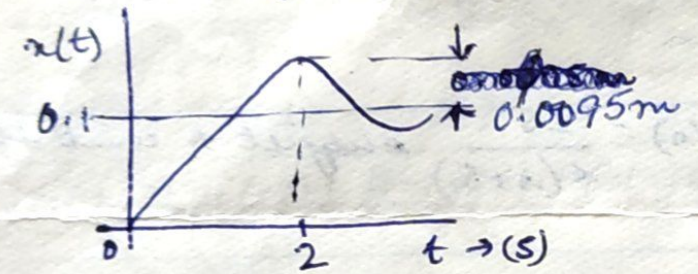






∴ desire high  $\omega_n$ .

Ex 4.



Spring-mass-damper system  
 Step force 2 N  
 S.S. displacement of M = 0.1 m  
 Obtain M, f, K.

$$(Ms^2 + fs + k)X(s) = U(s) = \frac{2}{s}$$

$$\therefore X(s) = \frac{2}{s(Ms^2 + fs + k)}$$

FINAL VALUE THEOREM

$$\lim_{t \rightarrow \infty} x(t) = \lim_{s \rightarrow 0} sX(s) = \frac{2}{k} = 0.1 \Rightarrow k = 20 \text{ N/m.}$$

$$\frac{0.0095}{0.1}$$

$$M_p = 0.0095 = \exp(-\zeta \pi / \sqrt{1-\zeta^2}) \Rightarrow \zeta = 0.6$$

$$\Rightarrow \frac{(\ln 0.095)^2}{\pi^2} = \frac{\zeta^2}{1-\zeta^2}$$

$$\Rightarrow \frac{(\ln 0.095)^2}{\pi^2 + (\ln 0.095)^2} = \zeta$$

$$t_p = \frac{\pi}{\omega_d} = 2 \Rightarrow \omega_n \sqrt{1-0.36} = \frac{\pi}{2} \Rightarrow \omega_n = 1.96 \text{ rad/s.}$$

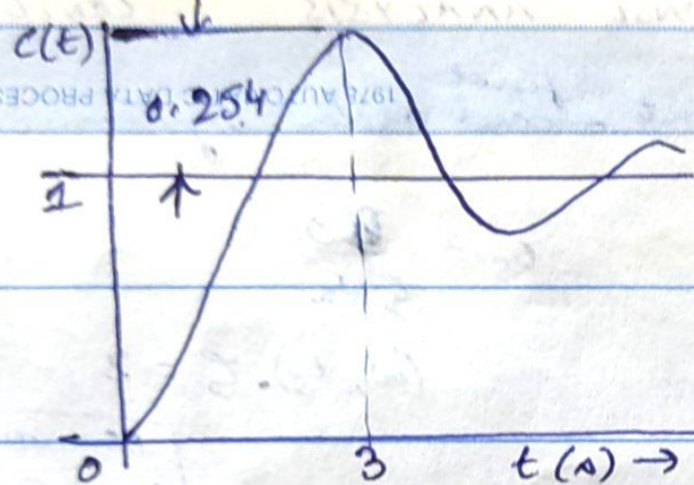
$$\omega_n = \sqrt{k/M} \Rightarrow M = \frac{20}{(1.96)^2} = 5.2 \text{ kg}$$

$$\zeta = \frac{1}{2} \sqrt{\frac{f}{KM}} \quad \text{OR} \quad 2\zeta\omega_n = \frac{f}{M} \Rightarrow f = \frac{M \cdot 2\zeta\omega_n}{M} = 12.2 \frac{\text{N}}{\text{rad/s}}$$

$$\zeta = \frac{\sqrt{\frac{(\ln M_p)^2}{\pi^2 + (\ln M_p)^2}}}{\frac{1}{2} \sqrt{\frac{f}{KM}}} \Rightarrow \left(\frac{\ln 0.095}{\pi}\right)^2 = \left[\frac{f^2}{K} + \left(\frac{\ln 0.095}{\omega}\right)^2\right] \zeta^2 \Rightarrow \left(\frac{\ln 0.095}{\pi}\right)^2 (1-\zeta^2) = \frac{f^2}{K}$$



Ex 2



Unity f/b system with

$$G(s) = \frac{K}{s(Ts+1)}$$

Unit step i/p.

Determine  $K, T, t_n, t_s$

$$M_p = 0.254 = \exp\left(-\pi \zeta / \sqrt{1-\zeta^2}\right)$$

$$\therefore \zeta = 0.4$$

$$t_p = 3 = \frac{\pi}{\omega_n \sqrt{1-\zeta^2}} \Rightarrow \omega_n = 1.142 \text{ rad/s.}$$

$$1+GH = 0 = Ts^2 + s + K = s^2 + 2\zeta\omega_n s + \omega_n^2$$

$$\therefore \omega_n^2 = \frac{K}{T} = (1.142)^2 ; 2\zeta\omega_n = \frac{1}{T} \Rightarrow T = 1.0946$$

$$\Rightarrow K = 1.42$$

$$\omega_d = 1.046 ; \theta = \tan^{-1} \frac{\sqrt{1-\zeta^2}}{\zeta} = 66.4^\circ$$

$$\therefore t_n = \frac{\pi - \theta}{\omega_d} = 2 \text{ s} ; t_s = \frac{4}{\zeta\omega_n} \text{ for } 2\% = 6.97 \text{ s} \quad \checkmark$$

$= 8T$



Ex3. Unity f/b control system with unit step i/p & DESIGN  
 $G(s) = \frac{K}{s(Ts+1)}$   $K, T$ : motor gain & time constants.

Fast transient desired with  $M_p < 5\%$ ,  $t_s < 4s$  (2%)

$$\therefore t_s = \frac{4}{\zeta \omega_n} \leq 4s \Rightarrow \zeta \omega_n \geq 1$$

$\therefore \zeta \omega_n = 1$  and  $\zeta = 0.707$  ( $M_p = 4.3\%$ ) satisfies requirement.

$$\therefore \omega_n^2 = 2$$

$$1 + GH = Ts^2 + s + K = 0 = s^2 + 2\zeta\omega_n s + \omega_n^2 \quad \text{CHARACTERISTIC EQN.}$$

$$\therefore \omega_n^2 = \frac{K}{T}, \quad 2\zeta\omega_n = \frac{1}{T} \Rightarrow T = 0.5s$$

$$K = 0.5 \times 2 = 1$$

Ex4. Unity f/b with Q TF  $G(s) = \frac{25}{s(s+5)}$  subject to unit step i/p

Find  $t_r, t_s, t_p, M_p$

$$s^2 + 5s + 25 = 0 = s^2 + 2\zeta\omega_n s + \omega_n^2 = 0$$

$$\therefore \omega_n = 5, \quad \zeta = 0.5. \quad \therefore \omega_d = 4.33, \quad \sigma = \zeta\omega_n = 2.5$$

$$\cos^{-1} \frac{1}{2} = \theta = \tan^{-1} \frac{\omega_d}{\sigma} = \tan^{-1} \frac{4.33}{2.5} = 1.047 \text{ rad} \quad \therefore t_r = \frac{\pi - \theta}{\omega_d} = \frac{\pi - 1.047}{4.33} = 0.482s$$

$$t_s = \frac{4}{\zeta\omega_n} (2\%) = 4s, \quad \frac{3}{\zeta\omega_n} (5\%) = 3s \quad 1.2s$$

$$t_p = \frac{\pi}{\omega_d} = 0.726s$$

$$M_p = \exp\left(-\frac{\pi\zeta}{\sqrt{1-\zeta^2}}\right) = 16.3\%$$

$$\frac{\sqrt{1-1/4}}{1/2} = \frac{\sqrt{3/4}}{1/2} = \frac{\sqrt{3}/2}{1/2} = \sqrt{3}$$

$$\theta = \tan^{-1} \frac{\omega_d}{\sigma} = \tan^{-1} \frac{\sqrt{0.75}}{0.5} = \tan^{-1} \frac{\sqrt{0.25 \times 3}}{0.5} = \tan^{-1} \sqrt{3} = \cos^{-1} \frac{1}{2}$$



## Impulse Response of 2<sup>nd</sup> order system:

$$R(s) = 1$$

$$\therefore C(t) = \mathcal{L}^{-1} C(s) = \mathcal{L}^{-1} \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

1 For  $0 < \zeta < 1$  UD

$$C(t) = \frac{\omega_n^2}{\sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \sin \omega_d t$$

2  $\zeta = 1$  CD  $C(t) = \omega_n^2 t e^{-\omega_n t}$

3  $\zeta > 1$  OD  $C(t) = \frac{\omega_n^2}{2\sqrt{\zeta^2-1}} \left[ e^{-(\zeta-\sqrt{\zeta^2-1})\omega_n t} - e^{-(\zeta+\sqrt{\zeta^2-1})\omega_n t} \right]$

## Ramp response

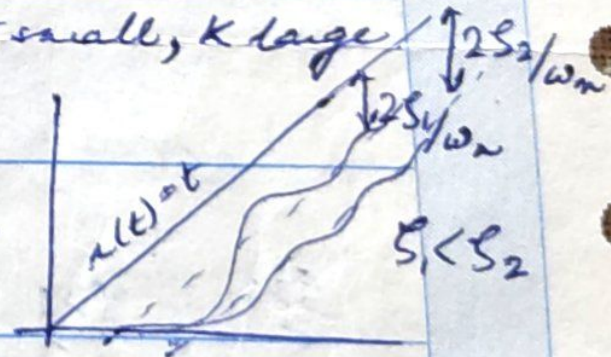
$$R(s) = 1/s^2$$

$$C(t) = t - \frac{2\zeta}{\omega_n} + \frac{e^{-\zeta\omega_n t}}{\omega_n \sqrt{1-\zeta^2}} \sin(\omega_d t + \theta)$$

$$e_{ss} = \frac{2\zeta}{\omega_n} = \lim_{s \rightarrow 0} sE(s) \Rightarrow \zeta \text{ not large, } \omega_n \text{ not small, } K \text{ large}$$

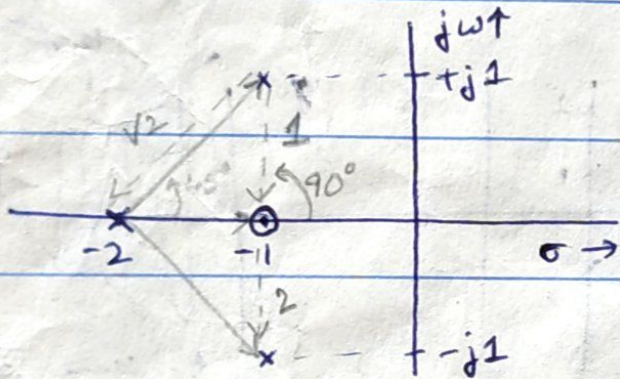
$$\frac{2\zeta\omega_n}{\omega_n^2} = b/k = \frac{b}{M} \frac{M}{K} \because 1+GH = (Ms^2 + fs + K) = 0$$

For lower  $\zeta$ , ramp response is more oscillatory.





# GRAPHICAL INTERPRETATION OF LAPLACE-HEAVISIDE EXPANSION:



$$G(s) = \frac{4(s+1)}{(s+2)(s^2+2s+2)}$$

$$= \frac{C_1}{s+2} + \frac{C_2}{s+1-j} + \frac{C_3}{s+1+j}$$

① towards origin  $\theta = 0$

$$C_1 = \frac{4(-2 - (-1))}{\sqrt{2} \cdot \sqrt{2}} = -2$$

② CCW from  $\sigma$  axis to  $+j\omega \rightarrow \theta +ve$   
from Z/p to test pt.

$$C_2 = \frac{4 \{ (-1+j1) - (-2) \}}{\{ (-1+j1) - (-2) \} \{ (-1+j1) - (-1-j1) \}} = \frac{4j}{(1+j1)(2j)} = 1-j1$$

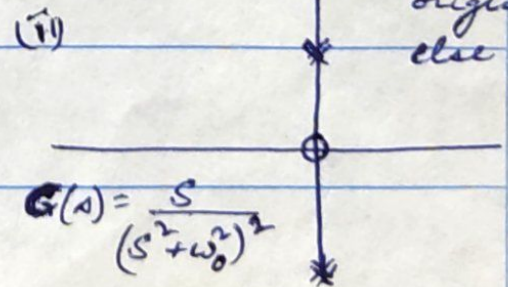
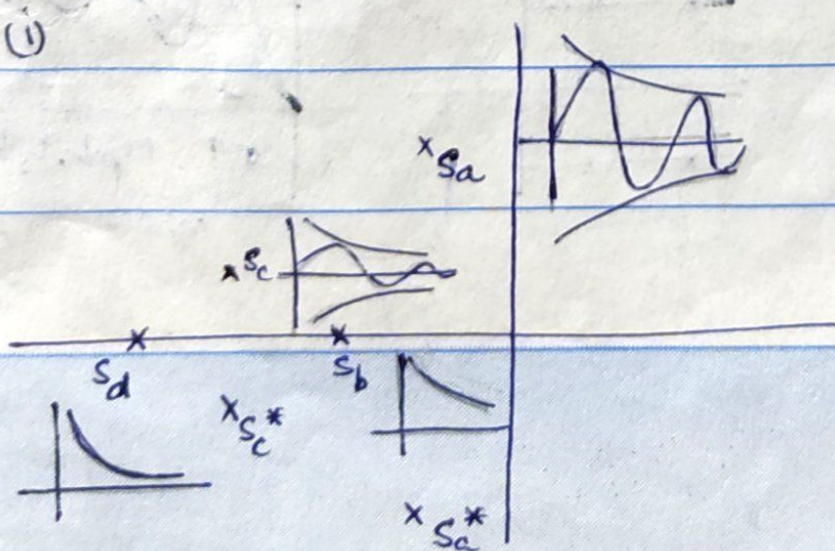
① tow.  $j\omega$  +ve  $\theta = 0$

$$= \frac{4 \cdot 1 e^{j90^\circ}}{e^{j90^\circ} 2 \sqrt{2} (e^{j45^\circ})}$$

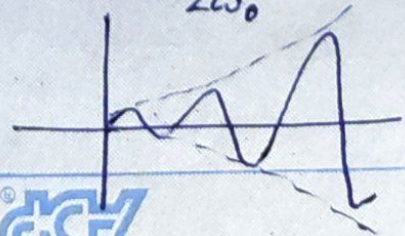
$$\sim \text{by } C_3 = 1+j1$$

## TIME DOMAIN BEHAVIOUR FROM POLE-ZERO PLOT

critical stable when  
ONLY SIMPLE POLE at  
origin  
else

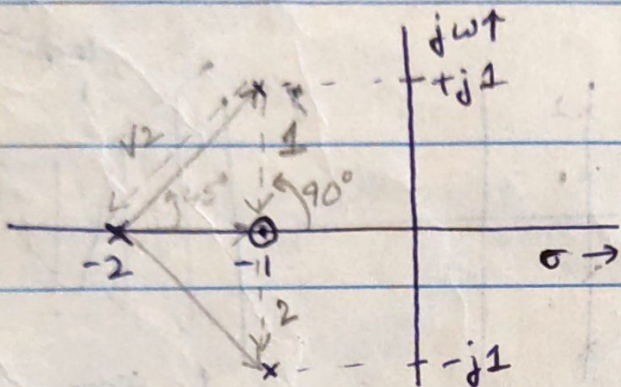


$$\therefore \phi(t) = \frac{t}{2\omega_0} \sin \omega_0 t$$





# GRAPHICAL INTERPRETATION OF LAPLACE-HEAVISIDE EXPANSION:



$$G(s) = \frac{4(s+1)}{(s+2)(s^2+2s+2)}$$

$$= \frac{C_1}{s+2} + \frac{C_2}{s+1-j} + \frac{C_3}{s+1+j}$$

① towards origin point  $\theta = 0$

② CCW from  $\sigma$  axis tow.  $+j\omega \rightarrow \theta +ve$   
from z/p to test pt.

$$C_1 = \frac{4(-2-(-1))}{\sqrt{2} \cdot \sqrt{2}} = -2$$

$$C_2 = \frac{4 \{ (-1+j1) - (-2) \}}{\{ (-1+j1) - (-2) \} \{ (-1+j1) - (-1-j1) \}} = \frac{4j}{(1+j1)(2j)} = 1-j1$$

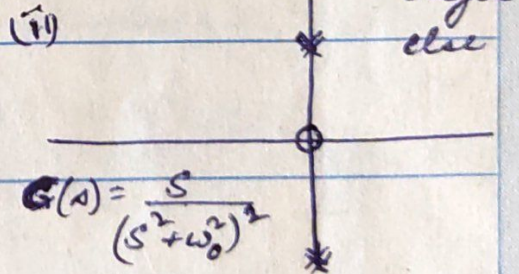
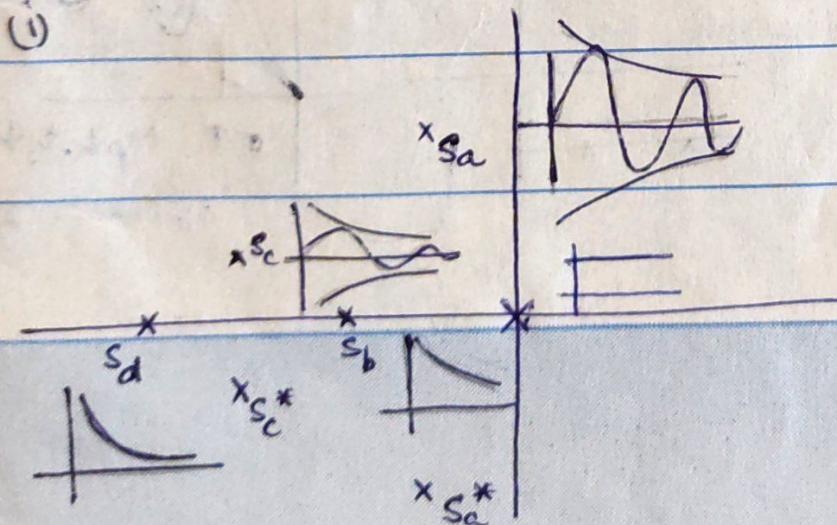
① tow.  $j\omega +ve \theta = 0$

$$= \frac{4 \cdot 1 e^{j90^\circ}}{e^{j90^\circ} 2 \sqrt{2} (e^{j45^\circ})}$$

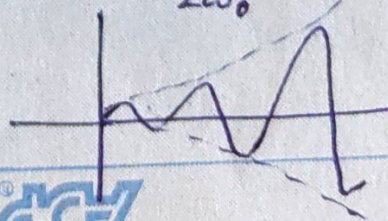
~ by  $C_3 = 1+j1$

## TIME DOMAIN BEHAVIOUR FROM POLE-ZERO PLOT

critical stable when ONLY SIMPLE POLE at origin else

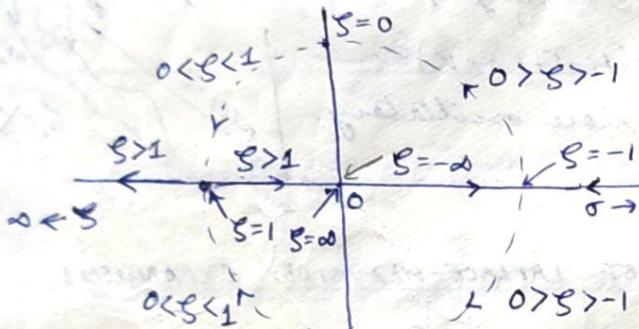
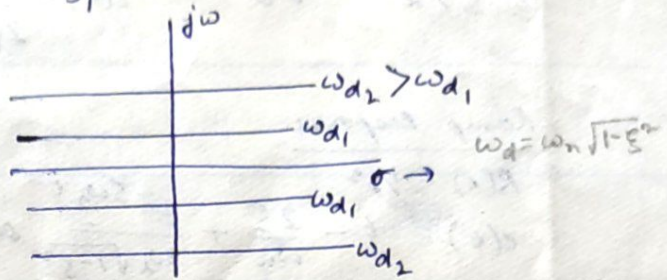
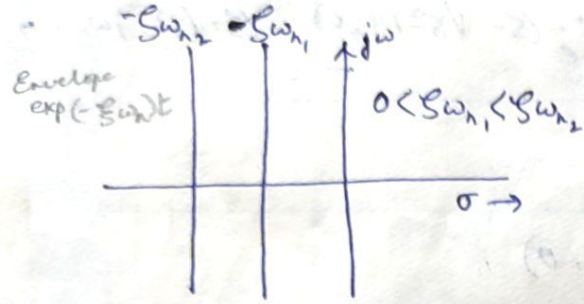
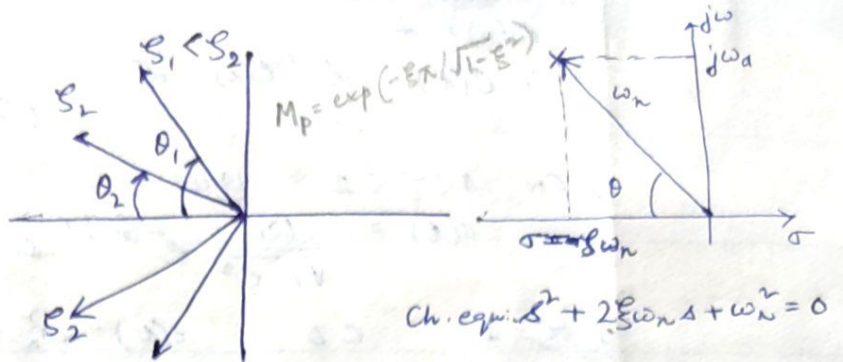
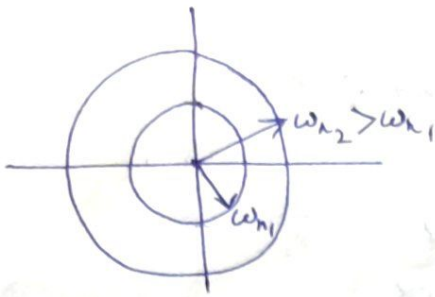


$$\therefore \phi(t) = \frac{t}{2\omega_0} \sin \omega_0 t$$

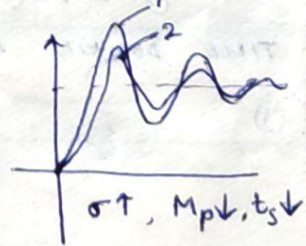
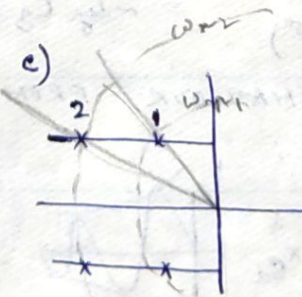
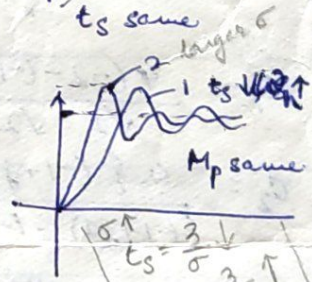
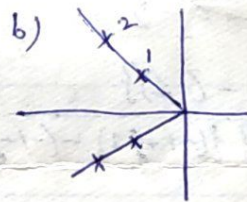
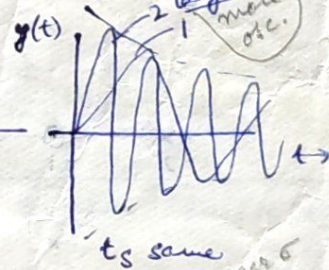
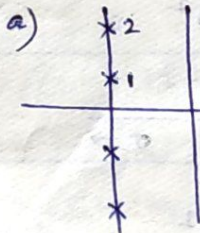
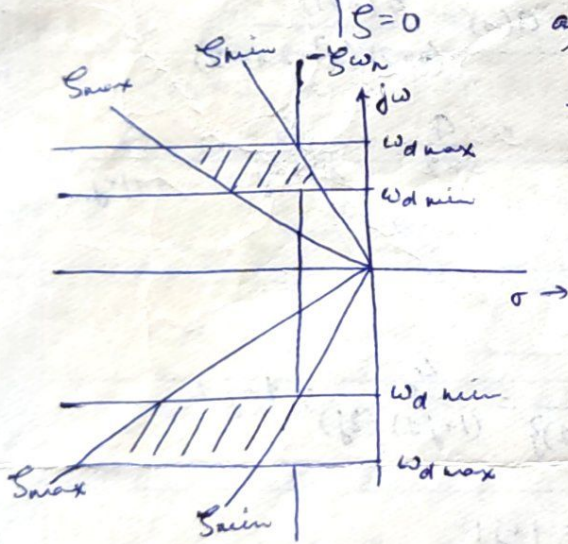




PERFORMANCE LINES



- $\zeta < 0$   $s_{1,2} = \pm \zeta\omega_n \pm j\omega_n\sqrt{1-\zeta^2}$   $\text{Re } s_{1,2} > 0$   
DIVERGING RESPONSE
- $\zeta = 0$   $\pm j\omega_n$  OSCILLATORY
- $0 < \zeta < 1$   $-\zeta\omega_n \pm j\omega_d$  UNDERDAMPED
- $\zeta = 1$   $-\omega_n$  C. DAMPED
- $\zeta > 1$   $-\zeta\omega_n \pm \omega_n\sqrt{\zeta^2-1}$  O.D.  $\text{Re } s_{1,2} < 0$



$\zeta \downarrow \Rightarrow \omega_d \uparrow \Rightarrow T \downarrow \Rightarrow$  faster osc.  
 $\omega_n \uparrow$

$\zeta \uparrow \Rightarrow M_p \downarrow$

$|\zeta| \uparrow \Rightarrow$  transient faster, envelope sharper

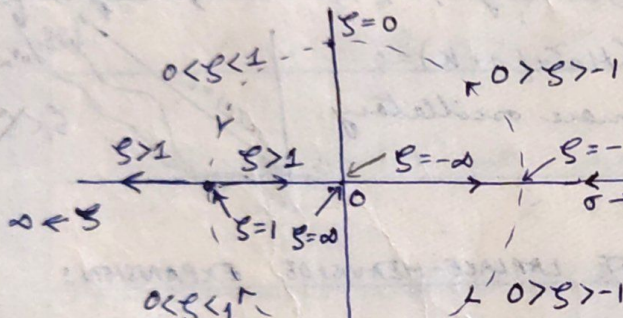
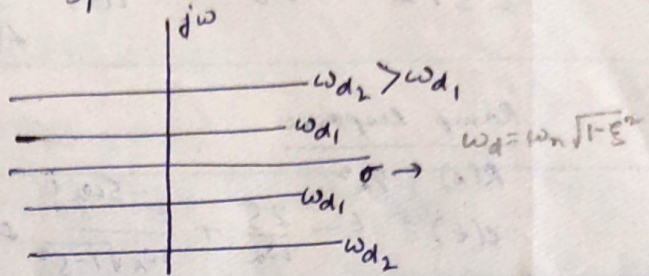
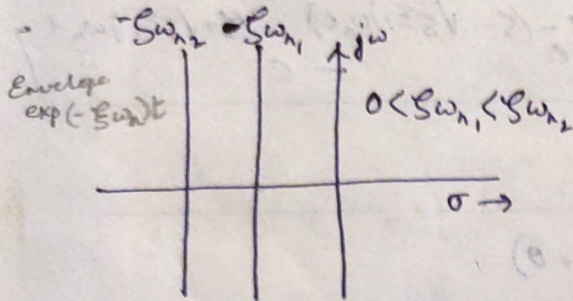
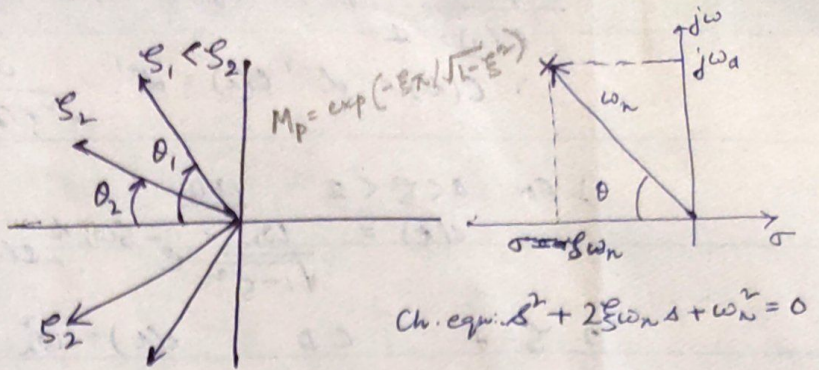
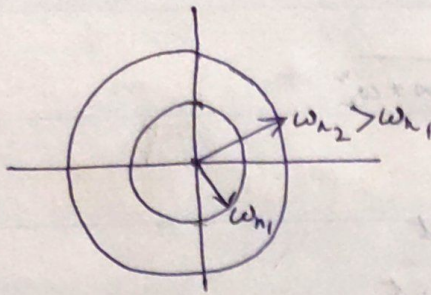
$\sigma$  more for 2,  $t_s = \frac{3}{\sigma} \downarrow$

also  $\zeta_2 > \zeta_1 \Rightarrow M_{p2} < M_{p1}$

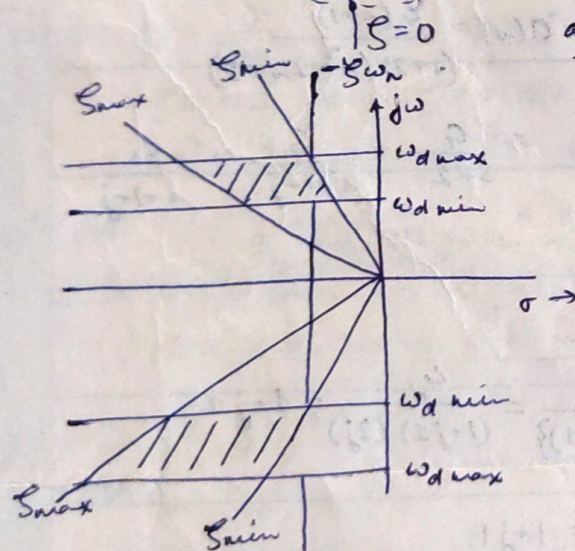
$\omega_d$  same  $\therefore$  period same.



PERFORMANCE LINES



1.  $\zeta < 0$   $s_{1,2} = \pm \zeta\omega_n \pm j\omega_n\sqrt{1-\zeta^2}$   $\text{Re } s_{1,2} > 0$   
DIVERGING RESPONSE
2.  $\zeta = 0$   $\pm j\omega_n$  OSCILLATORY
3.  $0 < \zeta < 1$   $-\zeta\omega_n \pm j\omega_d$  UNDERDAMPED
4.  $\zeta = 1$   $-\omega_n$  C. DAMPED
5.  $\zeta > 1$   $-\omega_n \pm \omega_n\sqrt{\zeta^2-1}$  O.D.  $\text{Re } s_{1,2} < 0$

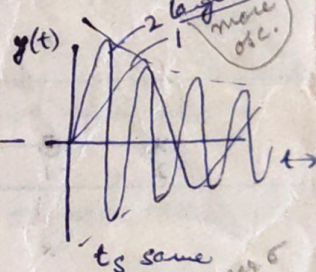
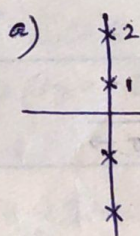


$M_p \rightarrow \zeta$ ;  $t_p = \pi / \omega_d$

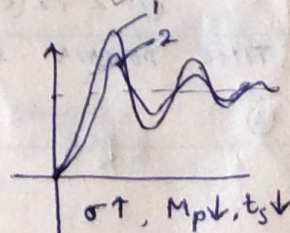
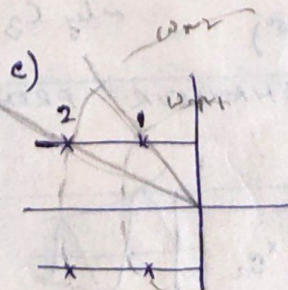
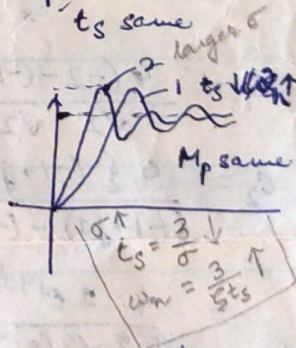
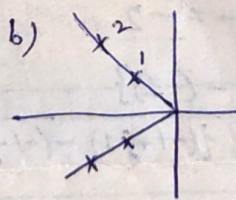
$\zeta \downarrow \Rightarrow \omega_d \uparrow \Rightarrow T \downarrow \Rightarrow$  faster osc.  
or  $\omega_n \uparrow$

$\zeta \downarrow \Rightarrow \zeta \uparrow \Rightarrow M_p \downarrow$

$\zeta \uparrow \Rightarrow$  transient faster, envelope sharper



envelope same.



$\zeta$  more for 2,  $t_s = \frac{3}{5} \omega_n \downarrow$   
also  $\zeta_2 > \zeta_1 \Rightarrow M_{p2} < M_{p1}$

$\omega_d$  same  $\therefore$  period same.