

M.E. MECHANICAL ENGINEERING FIRST YEAR FIRST SEMESTER EXAM 2025

SUBJECT: MECHANICAL VIBRATION

Time: 3 Hours

Full Marks: (20 X 5) = 100

Any missing data may be assumed with suitable justificationThe symbols/notations carry its usual meanings(Answer Any Five Questions)

Q1. (a) Write the differential equation of motion for the system shown in Fig Q1(a). Also, determine the natural frequency of the damped oscillation and the critical damping coefficient. Assume the link to be massless. (10)

(b) Find the Fourier series expansion of the periodic function shown in Fig. Q1(b). (10)

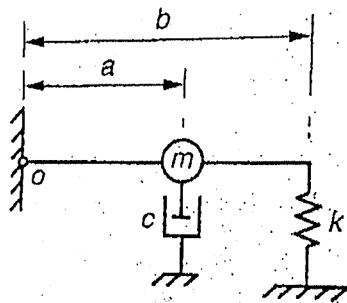


Fig. Q1(a)

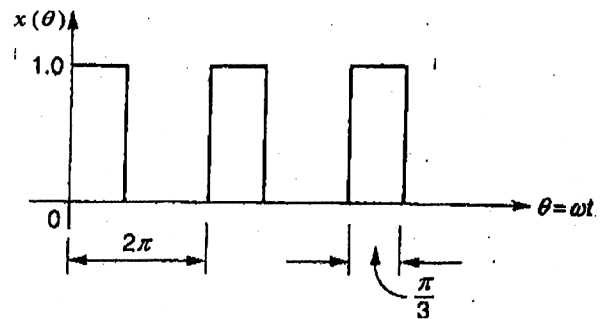


Fig. Q1(b)

Q2. (a) Fig. Q2(a) shows a mass block mounted on a spring and a Coulombic damper is oscillated under forcing function $F = F_0 \sin \omega t$. Prove that the steady state response amplitude of the system is,

$$X = \frac{F_0}{k} \left[\frac{1 - \left(\frac{4F_c}{\pi F_0} \right)^2}{\left\{ 1 - \left(\frac{\omega}{\omega_n} \right)^2 \right\}^2} \right]^{1/2}$$

(b) What is convolution integral? Using the integral, find the response history of the single-degree undamped oscillator for the forcing function, shown in Fig. Q2(b). (10)

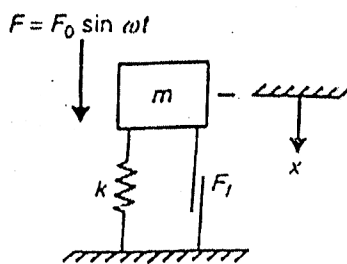


Fig. Q2(a)

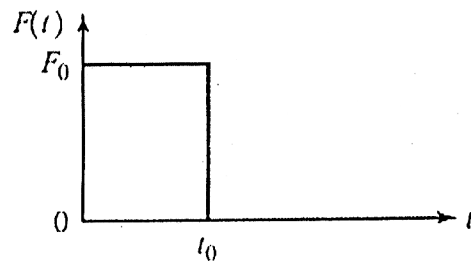


Fig. Q2(b)

Q3. Use static equilibrium to determine the stiffness matrix of the system shown in Fig. Q3.

(20)

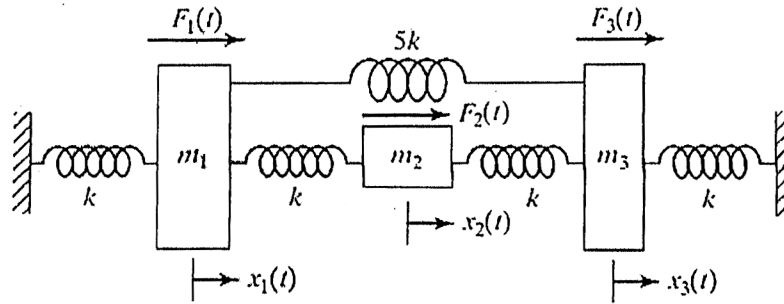


Fig. Q3

Q4. A machine tool, having a mass of $m = 1000$ kg and a mass moment of inertia $J_0 = 300$ kg-m², is supported on elastic supports, as shown in Fig. Q4. If the stiffnesses of the supports are given by $k_1 = 3000$ N/mm and $k_2 = 2000$ N/mm, and the supports are located at $l_1 = 0.5$ m and $l_2 = 0.8$ m, find the natural frequencies and mode shapes of the machine tool. (20)

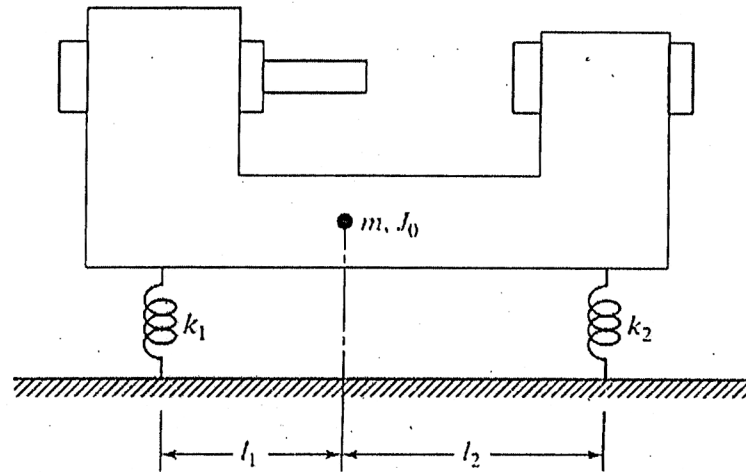


Fig. Q4

Q5. Using modal analysis, calculate the natural frequencies, mode shapes of the system whose equation of motion given below. Also, find the free-vibration responses of the individual masses of the system.

$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{Bmatrix} + \begin{bmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_2 + k_3 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

Assume the following data: $m_1 = 10$, $m_2 = 1$, $k_1 = 30$, $k_2 = 5$, $k_3 = 0$, and

$$\bar{x}(0) = \begin{Bmatrix} x_1(0) \\ x_2(0) \end{Bmatrix} = \begin{Bmatrix} 1 \\ 0 \end{Bmatrix}, \quad \dot{\bar{x}}(0) = \begin{Bmatrix} \dot{x}_1(0) \\ \dot{x}_2(0) \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix} \quad (20)$$

Q6. What is a string? Deduce the equation of motion of a string. Solve the equation to determine the natural frequencies and mode shape of a both ends fixed string. (20)

Q7. Prove the following relations in connection with the free transverse vibration of a beam with uniform cross-section:

$$(i) \quad EI \frac{\partial^4 w}{\partial x^4}(x, t) + \rho A \frac{\partial^2 w}{\partial t^2}(x, t) = 0$$

$$(ii) \quad \int_0^L W_i(x) W_j(x) dx = 0$$

(10 + 10)

-----X-----