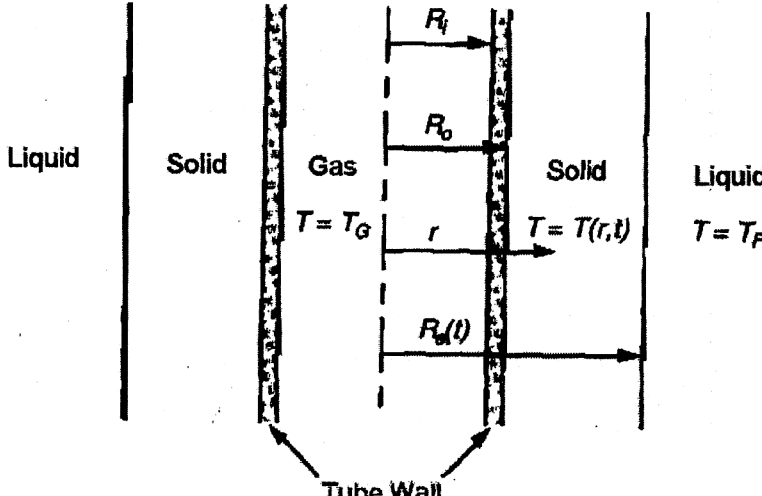
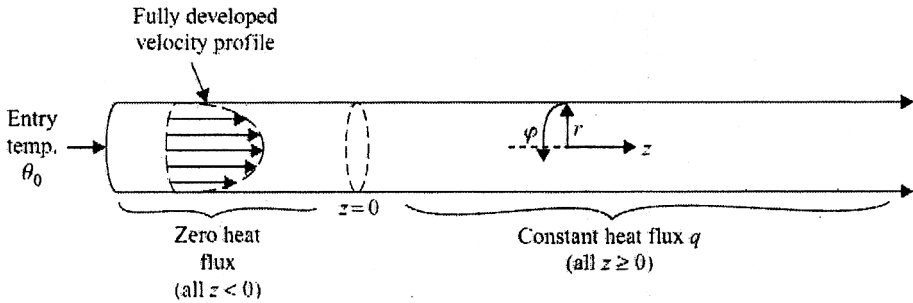


No. of Q/CO		Marks
1 (a)/ CO1	<p>Consider Fluid flow in porous media which is modelled by Darcy's law</p> $\mathbf{v} = -\frac{\kappa}{\mu}(\nabla P - \rho \mathbf{g}) \equiv -\frac{\kappa}{\mu} \nabla \mathcal{P},$ <p>Where κ is Darcy permeability (unit m^2) which is constant for a specific material. The microstructural details are not considered, Pore structure enters only through the value of κ, which is usually determined through experiments. (i) For a material of average porosity ε (pore volume fraction), derive the integral and differential form of conservation of mass (ii) Show that for an incompressible fluid :</p> $\nabla^2 \mathcal{P} = 0.$	(2+2)
1(b)/ CO2	 <p style="text-align: center;">Tube Wall</p> <p style="text-align: right;">FIG. 1</p> <p>A pure liquid is being frozen outside a refrigerated tube, as shown in Figure 1. The bulk liquid is at its freezing temperature (T_F) and the inner wall of the tube is cooled by convective heat transfer to a gas at bulk temperature T_G. The internal heat transfer coefficient is h and the heat of fusion is λ. Assume that freezing occurs slowly enough that the time derivative in the energy equation is negligible (pseudo steady state approximation) and T_G is independent of axial position. Assume that the tube wall is thin and has negligible thermal resistance.</p>	

No. of Questions/ CO		Marks
4.CO3 *	(i) Derive the laminar hydrodynamic boundary layer equations over a stationary flat plate of length L , considering U_∞ as the free stream velocity. (iii) Derive the integral momentum equations. (iv) Assuming that the nondimensional axial velocity is a fourth order polynomial of (y/δ) , derive the expressions for boundary layer thickness $\delta(x)$ and friction factor. (v) Using Reynolds averaging rules derive the Turbulent boundary layer equations over a stationary flat plate of length L . Define Eddy viscosity.	(6) (4) (12) (8)
5. CO3,CO4	 <p>FIG 3</p> <p>Consider that an incompressible Newtonian fluid at a uniform temperature θ_0 enters a circular tube of radius R. For $z > 0$ its wall receives a uniform heat flux q (refer to FIG.3). The bulk temperature T_b of the fluid increases as the fluid moves through the tube. Beyond a definite thermal entry length the shape of temperature profile does not change. This is called fully developed temperature profile.</p> <p>(i) Considering average fluid properties derive an expression of steady state fully developed laminar velocity profile through the tube. (8)</p> <p>(ii) Write the governing equation and boundary conditions for energy transport. Define the mixing cup temperature and derive the steady state fully developed temperature profile at far downstream. Prove that the value of Nusselt number is 4.364 (48/11). (12)</p>	(8) (12)

M. E. CHEMICAL ENGINEERING 1ST YEAR 1ST SEMESTER EXAMINATION 2025

SUBJECT: ADVANCED TRANSPORT PHENOMENA

Time: Three hours Full Marks 100

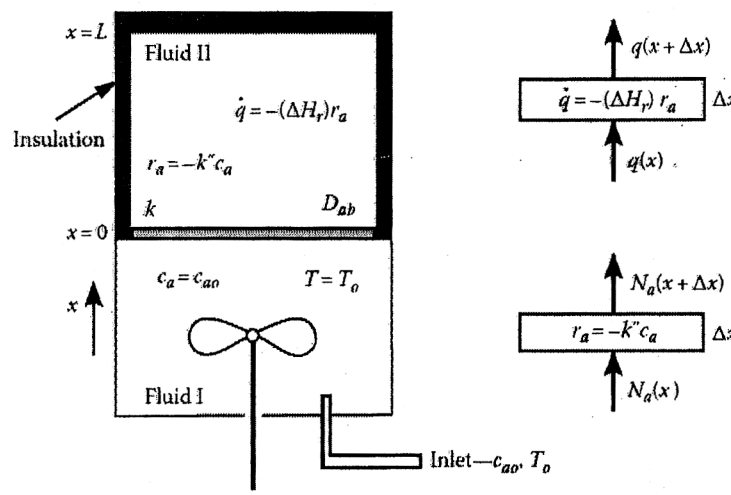
No. of Questions/ CO0		Marks
6. CO4	 <p>The diagram shows a cross-section of a vessel. Fluid I is at the bottom, well-mixed, with concentration $c_a = c_{a0}$ and temperature $T = T_0$. An inlet at the bottom right provides c_{a0}, T_0. Fluid II is at the top, stagnant, with a first-order reaction $r_a = -k''c_a$ and heat generation $\dot{q} = -(\Delta H_r)r_a$. The membrane between them has thermal conductivity k and diffusivity D_{ab}. The vessel walls are insulated. To the right, two differential balance boxes are shown for a slice of thickness Δx. The top box is for energy, showing heat flux $q(x)$ entering and $q(x+\Delta x)$ leaving, with a source term $\dot{q} = -(\Delta H_r)r_a \Delta x$. The bottom box is for mass, showing molar flux $N_a(x)$ entering and $N_a(x+\Delta x)$ leaving, with a sink term $r_a = -k''c_a \Delta x$.</p> <p>FIG. 4</p> <p>Two fluids are separated by a membrane permeable only to species 'a'. Fluid I is well mixed. The concentration of species 'a' is c_{a0} everywhere within this fluid, and the temperature in fluid I is held constant at T_0. Species 'a' diffuses through stagnant fluid II and undergoes a first order reaction ($a \longrightarrow b$) to form species 'b' within the fluid. Fluid I is replenished with 'a', while fluid II has an outlet (not shown) to remove excess 'a' and 'b'. The reaction is exothermic, yielding $-\Delta H_r$ joules per mole of 'a' reacted. The whole system is insulated from the surroundings, and all sides of the vessel are impermeable to both fluids. Write the governing balance equations and boundary conditions. Nondimensionalize the equations and name the dimensionless numbers. Derive the concentration distribution and temperature distribution in the system. Assume constant properties, dilute solutions (solute 'a' is sparingly soluble in fluid II) and the volume of fluid II remains constant.</p>	(20)

TABLE 2-1
General Conservation Equations for Interior Points and Interfaces

Interior points	Points at interfaces
$\frac{\partial b}{\partial t} = -\nabla \cdot \mathbf{F} + B_v$ (A)	$[(\mathbf{F} - b\mathbf{v})_B - (\mathbf{F} - b\mathbf{v})_A] \cdot \mathbf{n}_1 = B_s$
$\frac{\partial b}{\partial t} + \nabla \cdot (b\mathbf{v}) = -\nabla \cdot \mathbf{f} + B_v$ (C)	$[(\mathbf{f} + b(\mathbf{v} - \mathbf{v}_1))_B - (\mathbf{f} + b(\mathbf{v} - \mathbf{v}_1))_A] \cdot \mathbf{n}_1 = B_s$

Navier Stokes equation is given below

$$\rho \left[\frac{\partial \vec{V}}{\partial t} + \vec{V} \cdot \nabla \vec{V} \right] = -\nabla p + \rho \vec{g} + \mu \nabla^2 \vec{V}$$

Continuity and components of Navier Stokes equations for cylindrical coordinate are given below

$$\frac{1}{r} \frac{\partial (rv_r)}{\partial r} + \frac{1}{r} \frac{\partial v_\theta}{\partial \theta} + \frac{\partial v_z}{\partial z} = 0$$

$$\rho \left(\frac{\partial v_r}{\partial t} + v_r \frac{\partial v_r}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_r}{\partial \theta} - \frac{v_\theta^2}{r} + v_z \frac{\partial v_r}{\partial z} \right) = -\frac{\partial p}{\partial r} + \mu \left[\frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial (rv_r)}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_r}{\partial \theta^2} - \frac{2}{r^2} \frac{\partial v_\theta}{\partial \theta} + \frac{\partial^2 v_r}{\partial z^2} \right] + \rho g_r$$

$$\rho \left(\frac{\partial v_\theta}{\partial t} + v_r \frac{\partial v_\theta}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_\theta}{\partial \theta} + \frac{v_r v_\theta}{r} + v_z \frac{\partial v_\theta}{\partial z} \right) = -\frac{1}{r} \frac{\partial p}{\partial \theta} + \mu \left[\frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial (rv_\theta)}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_\theta}{\partial \theta^2} + \frac{2}{r^2} \frac{\partial v_r}{\partial \theta} + \frac{\partial^2 v_\theta}{\partial z^2} \right] + \rho g_\theta$$

$$\rho \left(\frac{\partial v_z}{\partial t} + v_r \frac{\partial v_z}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_z}{\partial \theta} + v_z \frac{\partial v_z}{\partial z} \right) = -\frac{\partial p}{\partial z} + \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_z}{\partial \theta^2} + \frac{\partial^2 v_z}{\partial z^2} \right] + \rho g_z$$

Ref. No. Ex/PG/ChE/T/111/2025

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Energy Equation

Cylindrical: $T = T(r, \theta, z, t)$

$$\frac{\partial T}{\partial t} + v_r \frac{\partial T}{\partial r} + \frac{v_\theta}{r} \frac{\partial T}{\partial \theta} + v_z \frac{\partial T}{\partial z} = \alpha \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial T}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 T}{\partial \theta^2} + \frac{\partial^2 T}{\partial z^2} \right] + \frac{H_v}{\rho C_p}$$

Species-balance-equation

Spherical: $C_i = C_i(r, \theta, \phi, t)$

$$\frac{\partial C_i}{\partial t} + v_r \frac{\partial C_i}{\partial r} + \frac{v_\theta}{r} \frac{\partial C_i}{\partial \theta} + \frac{v_\phi}{r \sin \theta} \frac{\partial C_i}{\partial \phi} = D_i \left[\frac{1}{r^2} \frac{\partial}{\partial r} \left(r^2 \frac{\partial C_i}{\partial r} \right) + \frac{1}{r^2 \sin \theta} \frac{\partial}{\partial \theta} \left(\sin \theta \frac{\partial C_i}{\partial \theta} \right) + \frac{1}{r^2 \sin^2 \theta} \frac{\partial^2 C_i}{\partial \phi^2} \right] + R_{vi}$$

Error Function Table

$$\text{erf}(x) = \frac{2}{\sqrt{\pi}} \int_0^x e^{-t^2} dt$$

x	Hundredths digit of x									
	0	1	2	3	4	5	6	7	8	9
0.0	0.00000	0.01128	0.02256	0.03384	0.04511	0.05637	0.06762	0.07886	0.09008	0.10128
0.1	0.11246	0.12362	0.13476	0.14587	0.15695	0.16800	0.17901	0.18999	0.20094	0.21184
0.2	0.22270	0.23352	0.24430	0.25502	0.26570	0.27633	0.28690	0.29742	0.30788	0.31828
0.3	0.32863	0.33891	0.34913	0.35928	0.36936	0.37938	0.38933	0.39921	0.40901	0.41874
0.4	0.42839	0.43797	0.44747	0.45689	0.46623	0.47548	0.48466	0.49375	0.50275	0.51167
0.5	0.52050	0.52924	0.53790	0.54646	0.55494	0.56332	0.57162	0.57982	0.58792	0.59594
0.6	0.60386	0.61168	0.61941	0.62705	0.63459	0.64203	0.64938	0.65663	0.66378	0.67084
0.7	0.67780	0.68467	0.69143	0.69810	0.70468	0.71116	0.71754	0.72382	0.73001	0.73610
0.8	0.74210	0.74800	0.75381	0.75952	0.76514	0.77067	0.77610	0.78144	0.78669	0.79184
0.9	0.79691	0.80188	0.80677	0.81156	0.81627	0.82089	0.82542	0.82987	0.83423	0.83851
1.0	0.84270	0.84681	0.85084	0.85478	0.85865	0.86244	0.86614	0.86977	0.87333	0.87680
1.1	0.88021	0.88353	0.88679	0.88997	0.89308	0.89612	0.89910	0.90200	0.90484	0.90761
1.2	0.91031	0.91296	0.91553	0.91805	0.92051	0.92290	0.92524	0.92751	0.92973	0.93190
1.3	0.93401	0.93606	0.93807	0.94002	0.94191	0.94376	0.94556	0.94731	0.94902	0.95067
1.4	0.95229	0.95385	0.95538	0.95686	0.95830	0.95970	0.96105	0.96237	0.96365	0.96490
1.5	0.96611	0.96728	0.96841	0.96952	0.97059	0.97162	0.97263	0.97360	0.97455	0.97546
1.6	0.97635	0.97721	0.97804	0.97884	0.97962	0.98038	0.98110	0.98181	0.98249	0.98315
1.7	0.98379	0.98441	0.98500	0.98558	0.98613	0.98667	0.98719	0.98769	0.98817	0.98864
1.8	0.98909	0.98952	0.98994	0.99035	0.99074	0.99111	0.99147	0.99182	0.99216	0.99248
1.9	0.99279	0.99309	0.99338	0.99366	0.99392	0.99418	0.99443	0.99466	0.99489	0.99511
2.0	0.99532	0.99552	0.99572	0.99591	0.99609	0.99626	0.99642	0.99658	0.99673	0.99688
2.1	0.99702	0.99715	0.99728	0.99741	0.99753	0.99764	0.99775	0.99785	0.99795	0.99805
2.2	0.99814	0.99822	0.99831	0.99839	0.99846	0.99854	0.99861	0.99867	0.99874	0.99880
2.3	0.99886	0.99891	0.99897	0.99902	0.99906	0.99911	0.99915	0.99920	0.99924	0.99928
2.4	0.99931	0.99935	0.99938	0.99941	0.99944	0.99947	0.99950	0.99952	0.99955	0.99957
2.5	0.99959	0.99961	0.99963	0.99965	0.99967	0.99969	0.99971	0.99972	0.99974	0.99975
2.6	0.99976	0.99978	0.99979	0.99980	0.99981	0.99982	0.99983	0.99984	0.99985	0.99986
2.7	0.99987	0.99987	0.99988	0.99989	0.99989	0.99990	0.99991	0.99991	0.99992	0.99992
2.8	0.99992	0.99993	0.99993	0.99994	0.99994	0.99994	0.99995	0.99995	0.99995	0.99996
2.9	0.99996	0.99996	0.99996	0.99997	0.99997	0.99997	0.99997	0.99997	0.99997	0.99998
3.0	0.99998	0.99998	0.99998	0.99998	0.99998	0.99998	0.99998	0.99999	0.99999	0.99999
3.1	0.99999	0.99999	0.99999	0.99999	0.99999	0.99999	0.99999	0.99999	0.99999	0.99999
3.2	0.99999	0.99999	0.99999	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000

